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CALORIMETRIC DETERMINATION OF THE THERMODYNAMIC PROPERTIES OF SATURATED WATER IN BOTH THE LIQUID AND GASEOUS STATES FROM 100 to 374° C

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ABSTRACT

A calorimetric determination has been made of the thermal properties of saturated water, in both the liquid and gaseous states, from 100° C (56 kg/cm²) to the close vicinity of the critical point, 374° C (225 kg/cm²). This new determination supplements a previous determination in this laboratory in the range from 0 to 270° C. With the new apparatus designed and built to withstand severe conditions of temperature and pressure, satisfactory measurements were possible up to within 1 degree of the critical point.

Special tests of the possibility of mixture of the liquid and vapor phases in evaporation experiments indicated freedom from this effect, except within 1 degree of the critical point. These tests included direct measurements of specific volume at 370° C, which confirmed the calorimetric results.

A compilation was made of the thermal properties of saturated water in the interval from 100 to 374° C, giving the vapor pressure and the specific volume, entropy, and enthalpy, or heat content, for both the liquid and gaseous states. This compilation is based on the results of the present measurements, supplemented by data covering the range from 100 to 270° C from the previous investigation. These latter data are in accord with those of the present investigation at the same temperatures.

A number of comparisons between these values and certain groups of similar data from other laboratories are shown graphically.

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I. INTRODUCTION

A program of steam research, sponsored in this country by the American Society of Mechanical Engineers and shared by institutions in other countries, has gone far toward extending and improving steam tables for use in the engineering field. The new data already obtained have been appraised and coordinated by international conferences, thus promoting greater accord among the working steam tables which have been compiled. In addition to fulfilling the original purpose in the service of steam-power engineering, the new data have a wide interest and value in other fields of science, such as chemical thermodynamics, for example.

As its share of the research program, the National Bureau of Standards undertook to determine certain thermal properties of saturated water and steam. The first portion of this project, covering measurements below 270° C (pressure 56 kg/cm²), was completed and published in 1930. Meanwhile, the use of higher temperatures and pressures in steam engineering has led to a need for extension of the data to higher temperatures, and, for that purpose, a new investigation was begun in this laboratory. Completely new calorimetric apparatus, designed to withstand the temperatures and pressures encountered in the critical region, was built, although, at the time, it was not known how near the critical temperature successful calorimetric measurements could be made.

One by-product of this calorimetric project was a new determination of steam-saturation pressures and a compilation of tables of this property. This compilation was offered as an adjusted appraisal of the available data with the object of encouraging uniformity in steam tables by supplying a factor needed in correlation of steam data.

The present report gives the results of a new series of calorimetric measurements which extend the range of determination of enthalpy of water and steam up to 374° C (225 kg/cm²). These new measurements also overlap and repeat the part of the earlier series of 1930 between 100 and 270° C.

Especial attention has been given to experiments to give assurance that error was not caused by failure to separate vapor completely from liquid in evaporation experiments. These special experiments included a few direct volumetric measurements as an additional verification.

II. GENERAL DESCRIPTION OF METHOD AND APPARATUS

The essential principles of the method used in the present series of measurements have been developed in previous publications [1].¹ The method has been applied in a previous series of calorimetric measurements [2] completed in 1930, extending up to a temperature of 270° C

¹ The figures in brackets refer to references listed at the end of the paper.

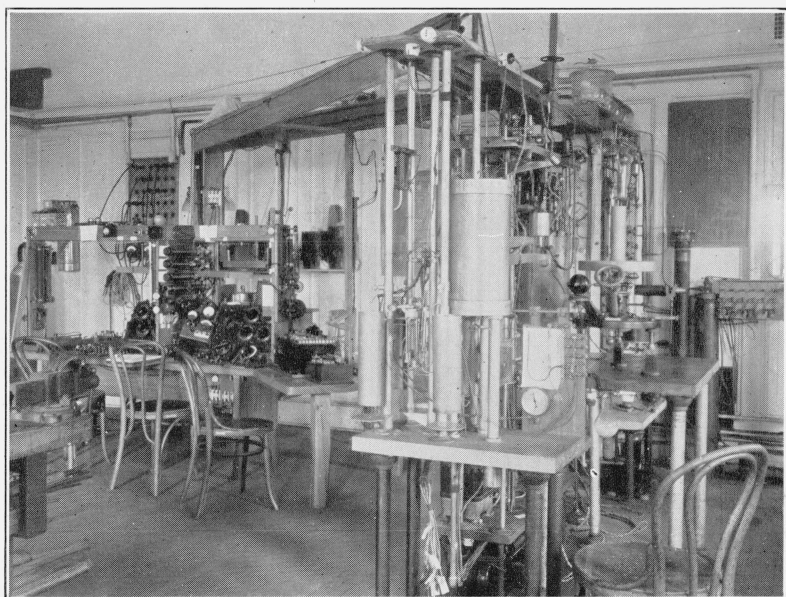


FIGURE 1.—*Assembled equipment.*

The electrical measuring instruments are on the left; the calorimeter is shown at the right.

(56 kg/cm²). For the higher temperatures and pressures to be encountered in the present investigation up to the critical region, completely new apparatus was built. Partial descriptions of the design of this apparatus and accounts of the progress of the investigation have been published in previous reports [3] from this laboratory. A view of the calorimetric apparatus and of the accessory electrical measuring instruments as set up in the laboratory is shown in figure 1.

Before attempting to describe details of design, construction, and operation, the general scheme of the experimental process will be outlined.

1. METHOD

The method makes use of a single calorimetric equipment with which measurements are made to determine several of the important thermal properties of the saturated fluid. A sample of water in a closed container at some chosen saturation state is heated electrically to some other chosen saturation state, or else is withdrawn either as saturated liquid or as saturated vapor. The energy required for each of these three processes is added and measured electrically. The amount of energy added per unit mass is characteristic of the particular experimental process observed. The group of measurements yields the enthalpy or heat content, together with several other important properties, and establishes the thermal behavior of the saturated fluid in the region covered by the survey.

2. APPARATUS

The apparatus consists essentially of a calorimeter which provides a place where a sample of water may be so isolated from other bodies as to enable its amount, state, and energy to be accounted for. The sample may be made to pass through a chosen, accurately determined change of state, while the accompanying gain or loss of energy is likewise accurately determined. The design of the apparatus provides for several such experimental processes selected for their physical simplicity and for their fitness to exhibit the thermal behavior of the fluid.

A quantity of water, part liquid and part vapor, is inclosed in a metal calorimeter shell. An electric heater immersed in the water provides a means of adding measured energy to the calorimeter system. Outlets with valves provide for putting in or taking out either liquid or vapor. Detachable receivers suitable for weighing are connected to the outlets to hold the samples of water transferred.

For confining the energy, the calorimeter is well insulated from the influence of external sources of heat. In operation, the temperature of an enveloping shell is kept very close to that of the calorimeter shell itself. The residual small amount of heat which passes by leakage to or from the calorimeter system is accounted for.

Means are provided for observing the following quantities:

- a. Temperature of the calorimeter and contents.
- b. Pressure in the calorimeter.
- c. Mass of fluid contents of the calorimeter.
- d. Energy added to the system.

The apparatus is designed to permit three special types of measurements to be made. In the ideal case of perfect manipulation and

control of experimental conditions these would consist essentially in observations of the following processes:

1. Heating with fixed amount of contents (heat-capacity determinations).
2. Isothermal expansion by adding heat, evaporating liquid, and removing *saturated vapor* (latent-heat experiments).
3. Isothermal expansion by adding heat, evaporating liquid, and removing *saturated liquid*.

3. RÉSUMÉ OF THEORY OF METHOD

By using the results of the published theory [1], the measured quantities may be expressed algebraically in terms of familiar thermal properties. The experimental determinations of heat capacity of a sample of water give as a result the increase in the quantity,

$$H - Lu/(u' - u)$$

over the observed range of temperature, in which H and L denote, respectively, in energy units, the enthalpy of saturated liquid and heat of vaporization per unit mass, and u and u' denote specific volume of saturated liquid and vapor, respectively.

In the latent-heat experiments we measure the energy which is added when water is evaporated and withdrawn as saturated vapor at constant temperature. This amount of energy per unit mass is equal to

$$L + Lu/(u' - u)$$

The ratio $u/(u' - u)$ is the fraction of a unit mass evaporated but remaining within the calorimeter when unit mass is withdrawn.

The quantity $Lu/(u' - u)$ is complementary both to the determinations of heat capacity and to the latent-heat experiments in the evaluation of H and L . This quantity is measured conveniently at any desired temperature by observing in supplementary calorimetric experiments of the third type the energy required to produce the necessary evaporation when a certain quantity of saturated liquid is withdrawn.

Thus the three different types of experiment yield values of three quantities referred to certain temperatures. These quantities,

$$H - Lu/(u' - u), L + Lu/(u' - u), \text{ and } Lu/(u' - u),$$

are denoted by the symbols, α , γ , and β , respectively. From these are derived the values of enthalpy of saturated liquid, H , the enthalpy of saturated vapor, H' , and the heat of vaporization, L . In addition to these three characteristic energy quantities, there also may be derived from the calorimetric data the entropies of the saturated liquid and vapor, and by use of the vapor-pressure data, obtained in separate measurements, specific volume and internal energy may be calculated.

The properties of saturated water and steam which may be derived from the calorimetric data, and the formulas by which these derivations are made are as follows:

$$H = \alpha + \beta$$

$$H' = \alpha + \gamma$$

$$L = \gamma - \beta$$

$$\phi = \int \frac{\alpha dT}{T^2} + \frac{H}{T}$$

$$\phi' = \phi + L/T$$

in which the following notation is used:

$$\left. \begin{aligned} \alpha &= H - Lu/(u' - u) \\ \beta &= Lu/(u' - u) = Tu \, dP/dt \\ \gamma &= L + Lu/(u' - u) = Tu' \, dP/dt \end{aligned} \right\} \begin{array}{l} \text{measured} \\ \text{quantities} \end{array}$$

H = enthalpy or heat content per unit mass of saturated liquid ($\epsilon + Pu$), referred to an arbitrary zero.

H' = enthalpy per unit mass of saturated vapor, referred to the same zero as H .

L = heat of vaporization per unit mass ($H' - H$).

u = specific volume of saturated liquid.

u' = specific volume of saturated vapor.

t = temperature on international centigrade scale.

T = temperature in centigrade absolute.

ϵ = internal energy per unit mass.

P = vapor pressure at saturation limit.

ϕ = entropy per unit mass of saturated liquid, referred to an arbitrary zero.

ϕ' = entropy per unit mass of saturated vapor, referred to the same zero as ϕ .

The notation used here is that of the previous paper [1] in which the theory of the method was developed. Its retention here is not by reason of any disapproval of the notation adopted by the American Standards Committee but is used to facilitate references to the previous work.

III. APPARATUS

The essential features of the new calorimetric equipment may be explained by reference to a schematic diagram, figure 2. The metal calorimeter shell C contains besides the water sample, an electric heater H_1 and a system of radial silver plates, D , for diffusing heat. This shell is suspended within a thermally controlled inclosure or envelope E , which shields it against heat exchange with the surroundings. Provision is made for withdrawal of either saturated vapor through the vapor throttle valve, VT , or saturated liquid through the liquid throttle valve, LT . The saturated vapor withdrawn passes through a shut-off valve, V , a receiver valve, RV , and is condensed and collected in one of the three receivers, WR , which may be detached at the unions U and weighed. The saturated liquid withdrawn is likewise collected in a receiver WR_1 . Provision is made for proper evacuation of all flow lines through vacuum valves, VV . During withdrawal through either the vapor or liquid throttle valve, control of pressure in the calorimeter is accomplished by hand regulation

of these valves, using the differential-pressure indicator *I* to guide the operator. The pressure is transmitted by water through the pressure-transmission cell, *PC*, and glass capillary to a balancing air pressure in air receiver *AR*. The water-air meniscus in the capillary at *I* shows when the pressures are balanced. Temperature measurements at numerous points on the calorimeter shell and its

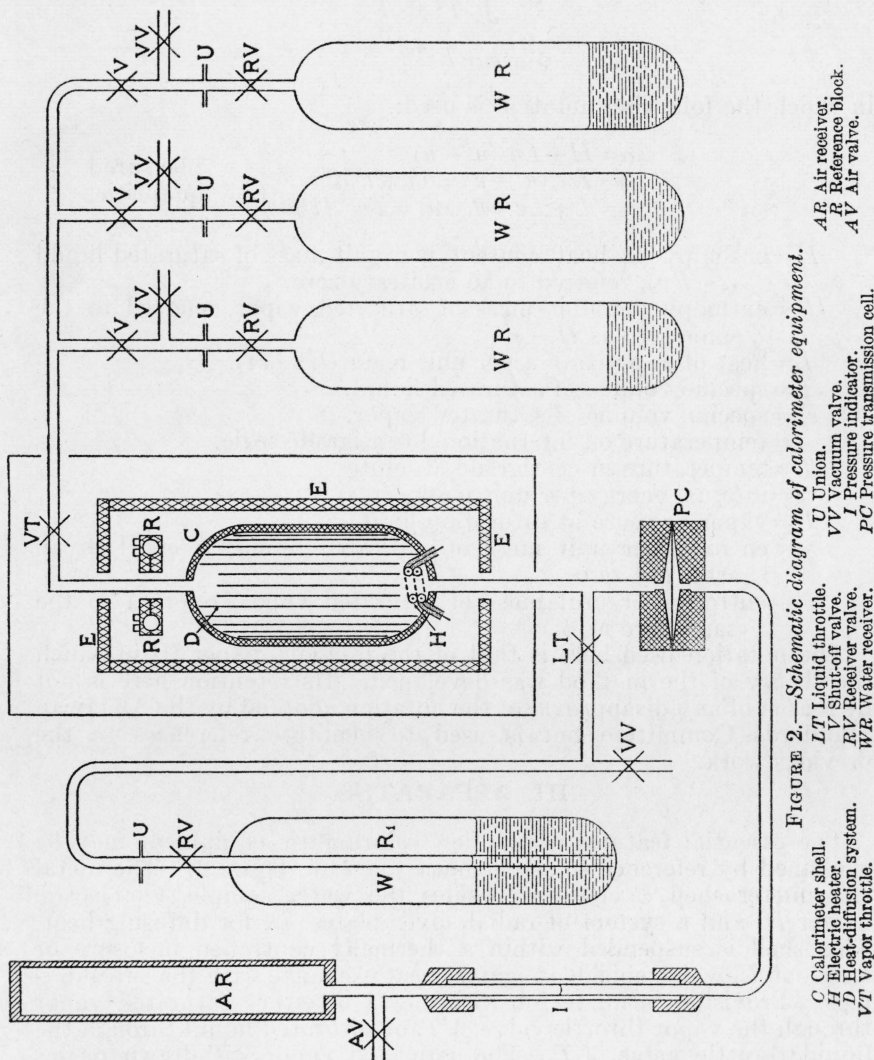


FIGURE 2.—Schematic diagram of calorimeter equipment.

C Calorimeter shell.
H Electric heater.
D Heat-diffusion system.
VT Vapor throttle.
LT Liquid throttle.
V Shut-off valve.
RV Receiver valve.
WR Water receiver.
PC Pressure transmission cell.
I Pressure indicator.
VV Vacuum valve.
R Reference block.
AR Air receiver.
AV Air valve.

surroundings are referred by means of numerous thermoelements to a "reference block", *R*, whose temperature is measured by means of two platinum-resistance thermometers. A scale drawing of the assembled apparatus is shown in figure 3. The principal parts will now be described.

AS—Adjusting screw.
H1—Calorimeter heater.
H2—Envelope heater.
H3—Reference-block heater.
H4—Guard heater.
H5—Tube heater.
H6—Tube and valve heater.
T—Platinum-resistance thermometer.
J1, J2—Principal thermoelement junctions.
WL—Liquid line.
VL—Vapor line.
WR—Water receiver.
PC—Pressure cell.
I—Pressure indicator (glass capillary).
AR—Air reservoir.

CW—Cooling water.
C—Calorimeter shell (special steel).
E—Envelope (silver).
S—Radiation shields (aluminum).
K—Outer casing (brass).
R—Reference block (silver).
D—Diffusion system (silver).
B—Baffle system (silver).
VT—Vapor throttle.
LT—Liquid throttle.
V—Shut-off valve.
RV—Receiver valve.
VV—Vacuum valve.
VM—Valve mechanism.

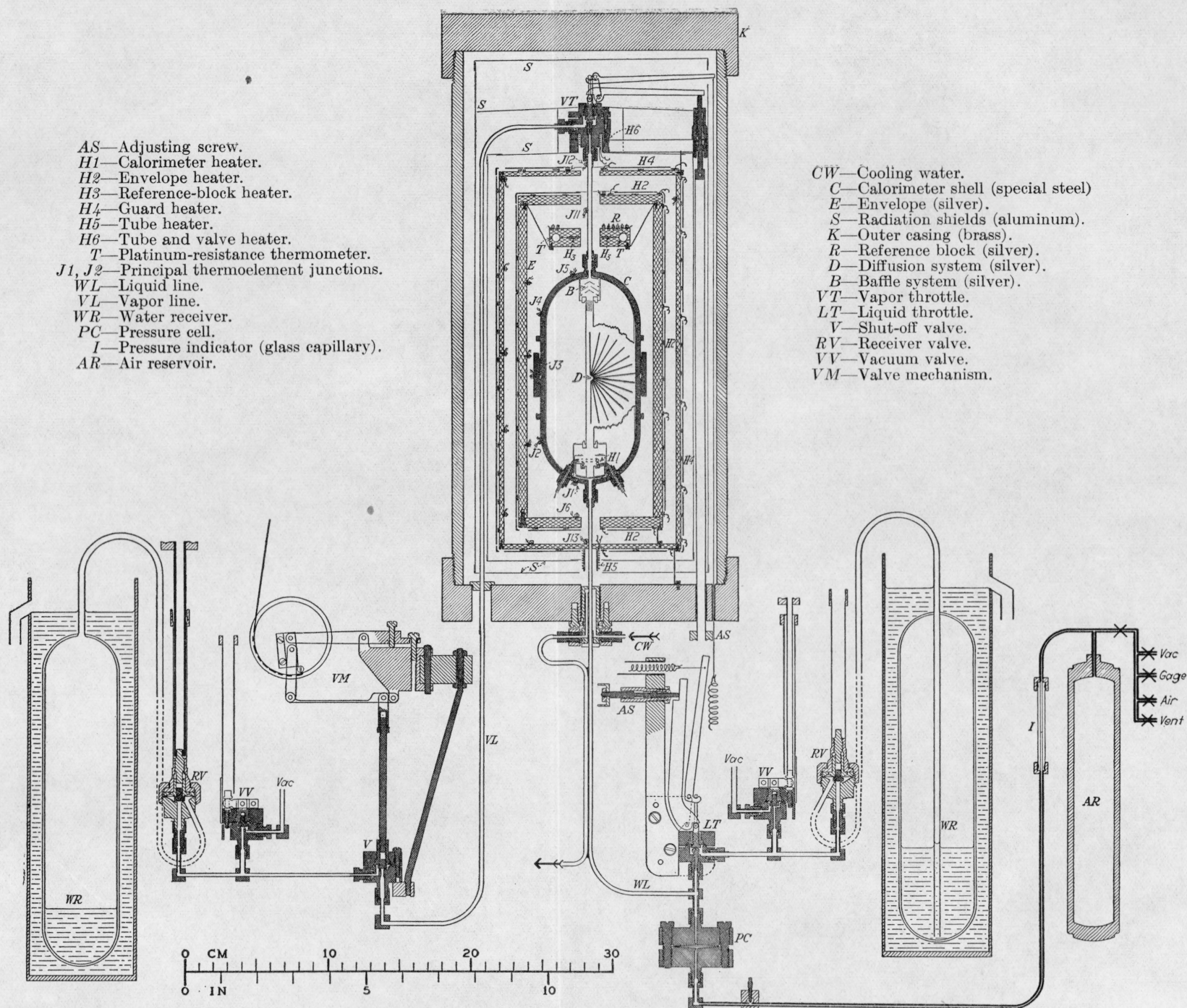


FIGURE 3.—Section of assembled calorimeter.

(Note: This is not a true section but shows important parts projected on a plane.)

1. CALORIMETER SHELL AND INCLUDED PARTS

The purpose of the calorimeter shell shown at *C* in figure 3 is to contain the sample of water while its thermal behavior is being observed. As assembled with the various accessory parts in place, it holds approximately 325 cm³. The material is a special alloy steel containing about 19 percent of Cr, 7.5 percent of Ni, 4.5 percent of W, 1.3 percent of Si, 0.5 percent of Mn, and 0.46 percent C. This material was chosen because of its resistance to creep and to attack by water at temperatures up to 400° C. The shell was machined from a solid bar and was made in two similar parts held together on a thin silver gasket by the tension developed in the right- and left-threaded band which is screwed on with powerful wrenches engaging machined lugs.

The shell, having a thickness of 3.2 mm, showed no permanent stretch with an internal hydraulic pressure of 4,500 lb/in². After carrying a charge of water at 300° C, the inner surface showed on examination no sign of attack by the water other than the formation of a very thin film of light straw color similar to that formed on the outside where in contact with air.

The electric heater in the calorimeter shell, shown at *H*₁ in figure 3, is an insulated resistor encased in a coiled metal tube sealed hermetically to the shell. It consists of about 10 ohms of calido (Ni, Cr, Fe alloy) wire, 0.2 mm in diameter, with gold leads. This resistor was wound in helical form, 0.6 mm outside diameter and embedded in magnesia for insulation. The magnesia was found to furnish adequate insulation for experiments at the high temperatures. The resistor was sheathed in a platinum tube drawn down tightly on the magnesia to an outside diameter of about 2 mm. This sheathed resistor was then bent to form two turns of a helical coil, 12 mm outside diameter. Each projecting end of this heater was sealed into a threaded plug of silver-palladium alloy by soldering with gold. To complete the seal to the calorimeter shell, the shoulders of these plugs were drawn tightly on thin silver gaskets by threaded nuts outside the shell.

Besides the two main leads which supply current to the heater, two other leads are provided for observing the potential drop in the heater. These potential leads are joined to the current leads midway between the calorimeter shell and the envelope in order to properly include the small amount of energy which is converted to heat in the current leads and transferred to the calorimeter.

The design of this heater was chosen to favor the calorimetric performance of the instrument. The heater was made small to avoid unnecessary heat capacity, to permit operation with small amounts of liquid, and to avoid excessive accumulation of energy as superheat in the liquid. The source of heat was purposely concentrated to cause active boiling and thus induce circulation of the water.

For promoting temperature equalization in the calorimeter shell, a system of heat-conducting plates was provided. A horizontal cross section is shown at *D* in figure 3. This heat-diffusing system consists of 30 flat plates of silver, 0.5 mm thick, shaped to conform closely to the vertical profile of the shell. These plates are held radially in two slotted hubs so as to penetrate and interconnect the space within the shell with a good heat conductor. The function of

this arrangement is to promote temperature uniformity throughout the calorimeter shell and contents.

A baffle system consisting of three conical sheets of silver-wire gauze with passages for steam staggered to prevent direct passage of water drops, is located immediately below the steam outlet as shown at *B*, figure 3. A view of the parts of the calorimeter shell prior to assembly is shown in figure 4.

2. PROTECTING INCLOSURE

The principal purpose of the inclosure which surrounds the calorimeter shell is to oppose heat leak to or from the calorimeter system and to provide for determining the correction for the small unavoidable heat leak. This inclosure consists of two coaxial cylindrical silver shells with flat ends, the inner shell designated as the "envelope" and the outer one the "guard." Pure silver was used for these shells on account of its high thermal conductivity and freedom from oxidation. The envelope is 6.3 mm and the guard 3.2 mm in thickness. Both these shells have electric heater elements properly distributed on their outside walls to permit raising their temperatures uniformly with the mean temperature of the calorimeter surface.

During periods when the calorimeter temperature is kept constant, no current is needed in the envelope heater, but sufficient current is supplied to the guard heater to furnish the heat lost to the outside and keep its mean temperature near that of the envelope and calorimeter. Under these conditions the envelope assumes a nearly uniform temperature, which can be adjusted very close to the mean surface temperature of the calorimeter. The guard temperature can be controlled automatically at any desired constant mean value by means of a sensitive thermoregulator actuated by a platinum-resistance thermometer installed on the inside wall of the guard.

Thermoelements with junctions distributed on the inner surfaces of envelope and guard indicate temperatures to guide the control of heat leak.

In the space outside the guard, two thin aluminum shields are placed to impede the loss of heat from the guard by radiation and convection. Since these shields are only 0.06 mm thick, they do not introduce any considerable lag. Outside the two shields a heavy brass casing, 7 mm thick, serves not only as a cover but also as a protection in case of an explosive failure.

3. CONNECTIONS TO THE CALORIMETER SHELL

The calorimeter shell is held in place by two tubes of silver-palladium alloy. The one at the top bears the weight of the shell and the one at the bottom centers it. Besides furnishing firm support for the shell without too great heat conduction, these tubes serve also for the transfer of fluid to or from the calorimeter. The upper one connects to the vapor-throttle valve, *VT*, which reduces the pressure and permits control of the rate of flow of outgoing steam in the gamma type of experiments. During the alpha experiments which preceded the gamma measurements, the upper tube was temporarily closed by a metal disk in the union at the top of the shell. This suspension tube extends to the calorimeter shell from the body of the vapor-throttle valve, *VT*, which is carried on a light but firm three-arm support from

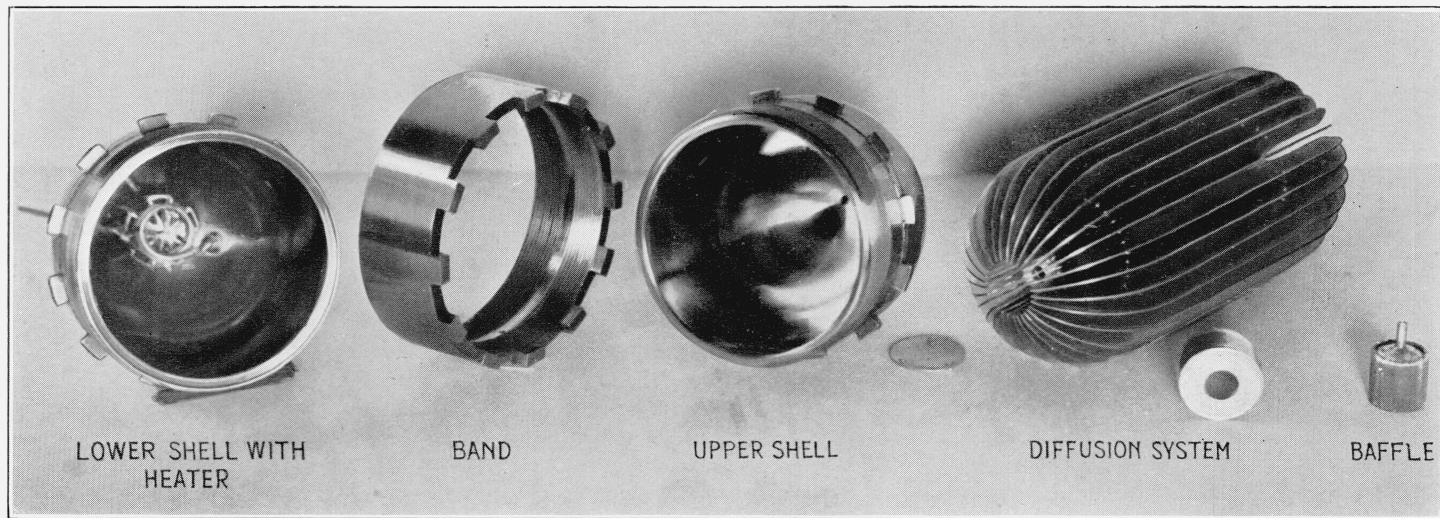


FIGURE 4.—*Unassembled calorimeter shell with accessory parts.*
(One-half actual size.)

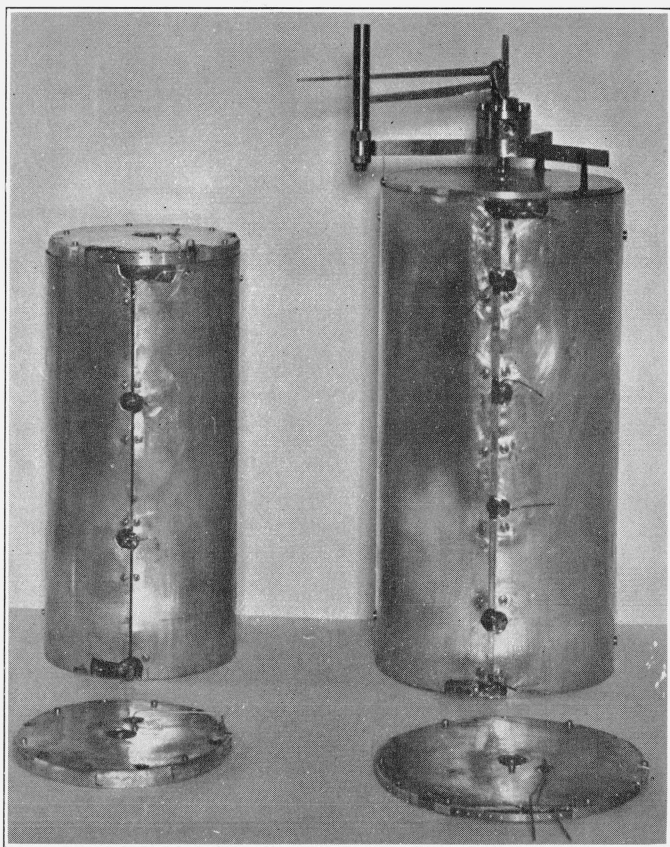


FIGURE 5.—Silver envelope shells.

(Envelope on left; guard with vapor throttle on right. One-fourth actual size.)

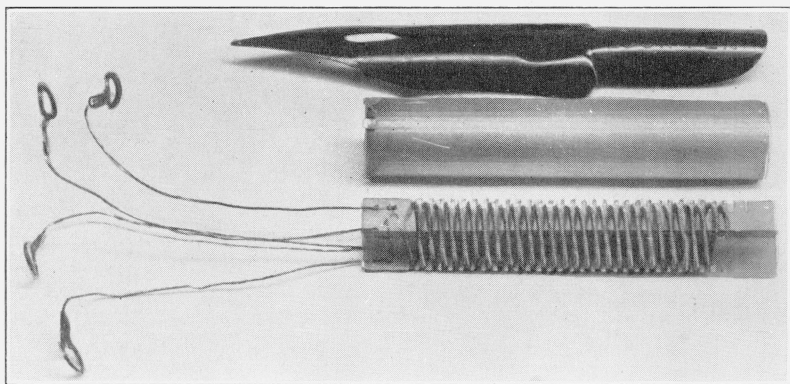


FIGURE 6.—Platinum-resistance thermometer and sheath, pen point for comparison of size.

the guard shell. The tube is of 3.0 mm outside diameter with a wall thickness of 0.5 mm.

The lower tube is 2.5 mm in diameter with a wall thickness of 0.4 mm. This tube extends down through a cooling water cell at the level of the brass casing, *K*. Just below this place, the tube ends in a union, not shown in figure 3. From this union an extension leads to a branch point. The two branches connect to the liquid-throttle valve, *LT*, and to the pressure-transmission cell, *PC*. The liquid-throttle valve is used both as a shut-off for the liquid line and also as a means of regulating the rate of flow of liquid in or out of the calorimeter. This valve and the line connecting to the receiver may be immersed in boiling water to drive off water trapped in these parts. The pressure-transmission cell, differential-pressure indicator, and liquid- and vapor-throttle valves will be more fully described later.

Heaters and thermoelements are provided to allow control of temperature gradients on both the upper and lower tubes. For the upper tube the heater is installed on the body of the vapor-throttle valve. The heater for the lower tube is mounted on a cylindrical silver support thermally attached to the tube where it passes through the silver guard shell. There is a free length of the tube of about 2.5 cm between this heater and the union at the calorimeter shell. Below this heater the tube is cooled by means of the water jacket, *cw*, which surrounds the tube at the level of the outer brass casing. The purpose is to establish a definite sharp gradient on the lower tube and also to help cool water when it is withdrawn hot from the calorimeter. It is important to avoid any danger of boiling in the lower tube during experiments, and it is also desirable that the temperature of the emergent portion of the lower tube containing water should not wander erratically.

From the liquid- and vapor-throttle valves, the flow lines lead to the water receivers, *WR*. These receivers are of pure silver of about 425-cm³ capacity, and are provided with outlet tubes, shut-off valves, *RV*, and unions to permit detaching the receivers for weighing. The receiver valves are of the diaphragm type with differential-thread mechanisms. To facilitate charging the calorimeter with a sample of water, two containers used for that purpose are provided with reentrant tubes reaching nearly to the bottom.

Beyond the vapor throttle the vapor flow line divides into three similar branches, each leading to an automatic shut-off valve, *V*. These valves and the lines immediately connected are immersed in an oil bath heated to a temperature which prevents condensation of vapor ahead of the valves. Experiments had shown that otherwise the distribution of the steam to the containers was erratic. The automatic mechanism for operating these valves by the time signal from the standard clock allows the flow of low-pressure steam from the throttle to be shifted instantaneously from one receiver to another without disturbance of the state of steady flow from the calorimeter. The technique of this operation will be more fully explained later under "experimental procedure."

The three vapor flow lines between the shut-off valves and receivers, and the liquid flow line between the liquid-throttle valve and the receiver, all have branches with valves, *VV*, so that all containers, flow lines, and the calorimeter may be evacuated. Also any small amount of water left after closing the receiver valve may be recovered by condensing in a detachable liquid-air trap in the vacuum line.

4. CONTROL VALVES

The valves used to handle and control the water as liquid and vapor are extremely vital parts. The operator's ability to keep control of the experimental conditions and also to account accurately for the amount of water depends on the nicety with which the valves may be adjusted for throttling or else closed entirely. No effort was spared in the design and workmanship of these valves to make them operate as perfectly as possible. In all of them a flexible silver diaphragm is used instead of packing to seal against the atmosphere and permit linear motion of the stem relative to the seat to open or close the flow aperture. The bodies, bonnets, seats, and clamping screws are all of stainless steel.

The two throttle valves used to control the flow either of vapor or of liquid from the calorimeter, are required to regulate the flow while throttling from the steam pressure down to the receiver pressure so that a steady energy input as heat is just compensated by evaporation of water and a steady state maintained. These differ from the shut-off valves principally in proportions and in the type of closing mechanism.

In all the control valves the aperture in the valve seat is closed by motion of the stem. The end of the stem is faced with an inlaid disk of metal, softer than the seat so that it can conform to the seat and seal tightly. The disk is renewable to allow for replacement when the sealing surface has been eroded. Silver and platinum have both been used for this part, the platinum having been found to resist erosion somewhat better than the silver. The stem has a travel of about 0.1 mm, which is accommodated by the flexibility of the silver diaphragm.

For fine adjustment of the throttle valve setting the stem is moved by end contact with a lever carried on a knife-edge fulcrum and driven by a slow-motion screw giving a large reduction in motion from the screw handle to the valve-stem end. A second lever actuated by tension of a spring takes up backlash or slack motion, and also permits still finer gradual adjustment of the valve setting by hand regulation of the spring tension, thus utilizing the elastic deformation of the valve structure to accomplish the ultimate adjustment.

The several shut-off valves not specially described are similar in principle to the throttle valves, except that the mechanisms used are nonautomatic and nonadjustable for regulation.

5. DIFFERENTIAL-PRESSURE INDICATOR

The differential-pressure indicator has played a very important part in the refinement of experimental technique. By its use, during the flow experiments in which vapor or liquid is withdrawn while evaporation goes on, visible motion of an air-water meniscus enables the operator to observe any slight deviation in pressure and thus allows him to adjust flow to maintain a practically constant pressure throughout the entire period of the experiment. This indicator is only part of a complete pressure-measuring apparatus, which has been described in a previous report [4].

The pressure is transmitted from the free surface of the liquid in the calorimeter through a continuous column of liquid water down to the thin elastic diaphragm of the pressure-transmission cell, *PC*.

The purpose of this cell is to allow the pressure in the calorimeter system to be transmitted to the indicator, I , while at the same time preventing the escape of water. Beyond the diaphragm another short water column extends to a balancing air pressure at a boundary meniscus in a glass capillary, I . This meniscus indicates the position of the diaphragm in the cell, and is used to show when the pressures on the two sides are balanced.

The transmitting diaphragm is a silver disk, 3 cm in diameter and 0.06 mm thick, stretched flat and clamped near the edge between the two parts of the cell. The diaphragm is most sensitive to changes of pressure when in its relaxed midposition. The cell walls allow a small displacement but never enough to deform the diaphragm beyond the elastic limit, even when the highest pressures used are applied to one side only. The total possible motion of the diaphragm allows a volume displacement of only about 0.04 cm^3 of water from the calorimeter.

The balancing air pressure in the glass capillary, I , is kept constant after initial adjustment, by means of an air reservoir, AR , immersed in an ice bath in a dewar vessel. The sensitivity of the indicator varies somewhat with the pressure, but, in general, a motion of the meniscus of 1 mm corresponds to a pressure change of about 0.001 atmosphere. The pressure indicator is effectually a sensitive thermometer responding quickly to small changes in temperature of the free evaporating surface in the calorimeter.

6. THERMOMETRIC INSTALLATION

The thermometric installation used in the control and measurement of temperature includes platinum-resistance thermometers and differential thermoelements. Two resistance thermometers installed in a silver reference block furnish a reference datum on the International Temperature Scale from which small temperature differences to the calorimeter or other points may be measured by means of thermoelements. Thermoelements alone are used for the survey of temperature distribution and for regulation.

The reference block was made of pure silver. It was designed to furnish adequate thermal connection between the resistance thermometers and the reference junctions of the thermoelements. The block is made of two similar rectangular pieces of silver, each 5.6 by 3.8 by 0.63 cm, held together flatwise by screws. It is suspended horizontally from the top of the envelope by four slender straps of stainless steel. A hole 12.7 mm in diameter in the vertical axis accommodates the small tube which suspends the calorimeter shell. The two resistance thermometers, T , and a pair of electric heaters, H_3 , fit snugly into appropriate receptacles machined across the horizontal midsection of the block. These heaters are provided for independent control of the temperature of the reference block.

Two specially constructed platinum-resistance thermometers were used as working standards for the temperature measurement. They were of the four-lead potential-terminal strain-free type described by C. H. Meyers [5].

The windings were made of pure platinum wound on a mica cross and placed in a silver sheath. The type of winding of these thermometers is shown in figure 6. Initial strains were relieved by annealing the completed thermometers at 660°C . The thermometers

were calibrated according to the specifications for the International Temperature Scale [6], which use the fixed points of ice, steam, and boiling sulphur as 0, 100, and 444.6° C, respectively. The thermometers fulfilled the requirements of the specifications for the International Temperature Scale. During the 4 years' use of the two thermometers, they have been calibrated several times and no significant changes were found. One thermometer was used regularly, while the second was used occasionally to check the first. The two thermometers, which were calibrated independently, agreed usually to better than 0.01° C within the temperature range of the experiments.

The thermoelements in the temperature-measuring system are used to survey temperature distribution, to indicate average temperature difference of surfaces, and to determine the temperature of the water sample. Of the 38 principal junctions of the thermoelements, 13 are

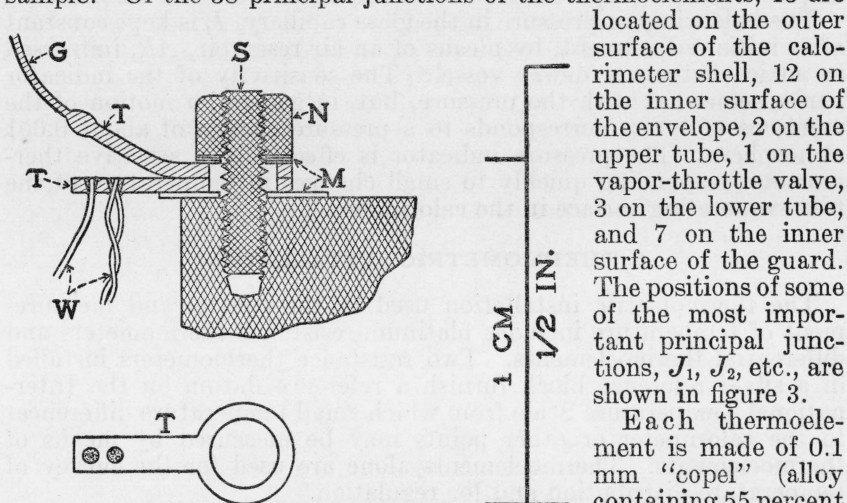


FIGURE 7.—Details of thermojunction attachment.

G Gold wire.	S Stud.
M Mica insulation.	T Gold terminal.
N Nut.	W Thermoelement wires.

located on the outer surface of the calorimeter shell, 12 on the inner surface of the envelope, 2 on the upper tube, 1 on the vapor-throttle valve, 3 on the lower tube, and 7 on the inner surface of the guard. The positions of some of the most important principal junctions, J_1 , J_2 , etc., are shown in figure 3. Each thermoelement is made of 0.1 mm "copel" (alloy containing 55 percent of Cu and 45 percent of Ni) and 0.16 mm "chromel P" (alloy containing 10 percent of Cr and 90 percent of Ni) wires, which are threaded through thin strips of mica for insulation and support. A principal junction is made by silver soldering the copel and chromel wires to a tiny gold terminal. Similarly, a reference junction is made by soldering either the copel or chromel to a gold terminal. Other gold terminals connect to gold lead wires. Figure 7 shows how the terminals are installed to secure good thermal connection,² electrical insulation, and electrical connection where desired. Additional thermal attachments of the gold lead wires are made at the reference block and at the bottom of the envelope to prevent thermal lead conduction from directly affecting the temperature of the reference junctions. The thermoelements used in the measurement of water temperature were calibrated in place against the resistance thermometers in the reference block. High accuracy in this calibration was not necessary

² A diagram showing the details of a thermoelement circuit similar to those used here is shown in a previous paper [2], page 244.

since the thermoelements were used to measure only very small temperature differences.

Various combinations of the thermoelements for the several functions mentioned above are provided by connections to specially built all-copper distributing switches.

By manipulation of these switches the observer can quickly shift from one to another combination, with only the delay for galvanometer response.

IV. EXPERIMENTAL PROCEDURE

In each of the three types of calorimetric experiment there are three principal quantities to be measured, namely, the amount of water subjected to the process, the amount of energy exchanged, and the change in state produced. It is the aim to account accurately for all three of these main factors, either by observations to evaluate separate elements, or by control measures to annul deviations from normal values. The following description of the experimental procedure indicates how these essential accountings were actually made.

1. ACCOUNTING FOR MASS OF WATER

The mass of water subjected to a change of state enters as a direct factor in the reduction of the data. The results, therefore, are no more reliable than the determinations of the masses. Furthermore, in any series of experiments made with a single charge in the calorimeter, a quantitative account of the total mass involved is important evidence concerning the reliability of the data derived from the individual experiments. Equality of the initial amount put in, to the amount taken out in several portions, indicates the absence of gross errors in the observing or recording of weights. Such a sum check was carefully kept in this work and when the accounts did not check properly, the results were discarded.

The general procedure employed in the preparation and handling of the water samples was the same in all types of experiments. A sample of water, purified by preliminary distillation for use in chemical analysis, was redistilled in a specially designed still under a pressure of about 0.1 atmosphere for the purpose of removing dissolved gases. These gases were withdrawn from the condenser together with some water vapor at such a rate that there was but little opportunity for the permanent gases to redissolve in the condensate. Special test experiments showed that this treatment was effective in reducing the residual gas left in the sample to less than 0.001 of the amount initially contained in distilled water. While this redistilled water was not strictly air-free, it was estimated to contain not more than 1/40,000,000 part of air by weight, or in other words, that the partial pressure of the residual dissolved gas in the water sample did not exceed 0.001 atmosphere. This purification was found adequate to avoid any partial pressure of air in the steam which could have a significant effect on the measurements. The purified sample of water was transferred to one of the silver water receivers, which had been evacuated to avoid recontamination with air during the transfer from the still to the calorimeter. Before connecting the water receiver to the water line for charging the calorimeter, the receiver was conditioned and weighed. The conditioning consisted in heating the container valve

to expel any water which was left in the external parts of the valve, then cooling the whole in a current of air from a fan, to the temperature of the room. This treatment was found to bring the receivers to a reproducible condition for weighing, and this routine technique was followed uniformly in loading and unloading the calorimeter in all types of experiments.

All weighings were made by the method of substitution. Corrections were made for buoyancy of the air on the calibrated weights. Correction for buoyancy on the silver container was avoided by using a counterpoise having approximately the same displacement as a container. In addition to the precautions enumerated above, an additional refinement was made in most of the later experiments where higher precision in weighing became more desirable. This refinement was to weigh a dummy silver container at nearly the same time as the silver container containing the sample of water. Since the dummy container was treated in much the same way as the container having the sample of water, the above procedure tended to eliminate possible errors due to changes in condition of surface of the container, changes in the balance arm, and uncompensated changes in air buoyancy.

After weighing, the receiver with its charge of water was attached by a union to the line leading to the bottom of the calorimeter. The calorimeter and line were then evacuated through the vacuum valve until the McLeod gage indicated a pressure of less than 0.001 mm. The system was then considered tight and sufficiently gas-free. The vacuum valve was then closed and the receiver valve opened. The water in the receiver was forced into the calorimeter by its own vapor pressure, using a gas torch to heat the receiver. After the liquid had been driven from the receiver, the calorimeter valve, *LT*, was closed, the valves surrounded with boiling water and the receiver immersed in cold running water. The water remaining in these connecting lines was thus distilled back into the receiver, which was then shut off and detached. Experiments showed that after this operation the vapor left in the lines was negligible in amount.

The receiver was then conditioned and weighed again to give the tare for determining the net charge introduced into the calorimeter system.

After a series of experiments in which part of the charge of water may or may not have been withdrawn, the water remaining in the calorimeter was withdrawn into the container. This was done while the calorimeter was hot, thus leaving the calorimeter and its outflow tube dry. After distilling the remainder over to the container and closing the container valve, the residue of vapor still left in the calorimeter could be recovered by condensing in a liquid air trap in the vacuum line. It was found, however, that this residue was very small and so uniform in amount that it need not be determined for each emptying. The final comparison of the masses in and out of the calorimeter was found to be an adequate check on the technique of determining the mass of water.

2. ACCOUNTING FOR ENERGY

Measured energy was supplied electrically to the calorimeter and contents by means of the calorimeter heating coil installed within the calorimeter shell and immersed in the water sample. Power was supplied by a separate 30-volt storage battery of large current capacity. A Wolff-Diesselhorst potentiometer was used for measurements of

current and potential drop in the heater. A 0.01-ohm four-terminal resistance in series with the heater was used for the current determination, and a ratio volt box for the potential drop. Correction was made for the fraction of the main current shunted through the volt box. In order to avoid too rapid initial change when the current was switched to the calorimeter heater at the start of an experiment, a dummy or spill resistance was used to steady the battery output. The shift was made instantaneously by means of a double-pole double-throw switch which was automatically actuated by the electric seconds signals furnished by the standard clock. The heating periods were usually an integral number of minutes and there was no significant error in the time measurement (probably less than $1/50,000$).

In one type of the gamma experiments, the time signals were used in a similar manner to actuate the shut-off valves which shifted the flow of steam from one receiver to another while the electric power input to the heater was uninterrupted. The energy supplied to the calorimeter while a given sample was being withdrawn was then calculated for the period thus determined.

All electrical measuring instruments were carefully calibrated at this Bureau. The potentiometric measurements of power input were referred to any one of a group of three cadmium standard cells. These cells were of the saturated type and were maintained at a constant temperature in a special temperature-control box described by Mueller and Stimson [7]. Frequent calibration of the standard cells proved their reliability.

In an ideal calorimetric experiment, only the energy supplied electrically to the calorimeter would need to be considered. In practice, the ideal of absolute thermal isolation of the calorimeter from the surroundings is never quite possible. In these experiments the thermal leakage was not negligible, yet it was made small by instrumental and experimental precautions, and this remaining heat transfer, which was usually about 0.01 percent of the total energy, was determined with the necessary precision for use as a correction term.

The actual heat-leak correction was evaluated with the aid of numerous differential thermoelements which indicated temperature differences between points on the calorimeter and points on its immediate surroundings. These temperature difference indications were observed every minute and summed algebraically to give integrated heat-leak factors. These factors when multiplied by heat-leak coefficients determined in special experiments, gave the heat-leak corrections. In these special experiments the temperature change in the calorimeter and contents was observed when the surroundings are controlled for a definite period of time at a known exaggerated temperature difference from the calorimeter. Knowing the approximate heat capacity of the calorimeter and contents, its temperature change could be reduced to actual heat transferred, and the heat-leak coefficients evaluated. In this investigation, the heat leaks are classified into four types. They are designated as "envelope heat leak", "top heat leak", "bottom heat leak", and "residual heat leak."

The envelope heat leak occurs when the envelope, or metal surface immediately surrounding or opposed to the surface of the calorimeter, differs from it in temperature. Twenty-three thermoelements distributed appropriately on these surfaces integrated their surface

temperatures and when properly connected indicated the average temperature difference.

In actual experimental conditions, the average temperature of the envelope could be regulated very close to that of the calorimeter shell. This nice temperature regulation of the envelope was effected directly in the first or alpha type of experiment (heat-capacity determinations) by use of the electric heater on the envelope. In the constant-temperature experiments of the second and third types, the regulation was maintained by automatic control of the temperature of an outer shell or "guard."

The top heat leak is due primarily to the heat conduction along the tube connected to the top of the calorimeter. After piercing the disk in this tube to provide for the gamma experiments, it was necessary to keep this tube slightly warmer than the calorimeter in order to avoid the possibility of condensation of water and transfer of unmeasured heat from the calorimeter system. This caused a small flow of heat toward the calorimeter, which was evaluated in terms of the temperature gradient in the tube as indicated by thermoelements. Temperature control of the upper part of the tube was effected by use of a special electric heater in two sections attached to the vapor-throttle valve. This heater also provided for compensating the cooling which occurred in the throttle valve when reducing the pressure of flowing steam in latent-heat experiments.

The bottom heat leak is analogous to the top heat leak with the difference that the lower tube is kept colder than the calorimeter to avoid the possibility of boiling in the lower tube with transfer of unmeasured heat to the calorimeter. It was evaluated in like manner, and the temperature of the lower part of the tube was controlled by a special electric heater, thermally attached.

The fourth type of heat leak is called "residual" because it still exists when the other three have been annulled. In other words, it is a measure of the heat gained or lost by the calorimeter when every known precaution against such a heat transfer has been taken. The value of this heat-leak rate was determined by means of blank experiments in which no electric energy was supplied. The other three heat leaks were controlled and corrected for. During the course of the gamma and beta measurements, about 300 separate determinations were made of this residual heat-leak rate. A study of these determinations by a semigraphic analysis showed that, while this rate was so small as to be nearly insignificant, yet it was definite enough to warrant applying corrections. Although the cause of this residual heat leak was unknown, these corrections were applied to all gamma and beta measurements to remove, as far as possible, this source of systematic error.

All four types of heat leak varied with temperature, and the coefficients were checked frequently during the experiments throughout the range of temperature.

3. ACCOUNTING FOR CHANGE IN STATE

Two means are available for evaluating the change in state of the sample of water, namely, measurement of either temperature or pressure. Either is suitable as an independent coordinate for expressing the properties of saturated steam, and convenience determines the choice practically.

Independent measurements have now established the characteristic relation between temperature and saturation pressure of steam with satisfactory accuracy in the range of the present measurements. Pressure observations were therefore unnecessary in the calorimetric measurements, except as they were used to indicate the state at which evaporation proceeded and thus enable the operator to control that process by hand regulation of the throttle valves. For determining the initial and final state of water in the calorimeter, temperature measurement was used entirely.

The temperature of the water in the calorimeter, when in equilibrium, was measured by the two platinum-resistance thermometers, used in conjunction with thermoelements, as previously described. The temperature change of the calorimeter and contained water sample was thus accurately observed in all experiments. All observed temperatures are expressed on the International Temperature Scale.

For determining either the initial or final temperature, each temperature reading consisted in simultaneous observations of resistance thermometer and thermoelements. The five thermoelements having principal junctions designated J_1 to J_5 , inclusive, were connected in series to indicate the mean temperature of the calorimeter with respect to the reference block. Four successive temperature readings were made at equal time intervals. This method of reading several successive temperatures is desirable for several reasons. The four-lead potential-terminal resistance thermometer requires at least two observations to eliminate the lead resistance from the measurement. Increasing the number of readings decreases the accidental error of observation. A regular schedule of readings takes account of slight drift of temperatures.

Provision is made for observing by differential thermoelements any deviation in temperature of steam or water withdrawn, from the initial saturation temperature, so that correction could be made, if required, for this change of state.

In actual experiments it was found that the steam left the calorimeter at the saturation temperature as closely as could be observed, so no correction need be applied. In the beta experiments when liquid water was withdrawn it was found that the correction was appreciable, although small.

In both the gamma and beta experiments, the first where vapor is withdrawn, the second where liquid is withdrawn, the control of the pressure to constancy is an important part of the accounting for the state of the fluid. The technique of this operation deserves special mention. The differential-pressure indicator (previously described) was extremely sensitive, particularly at the higher temperatures for which this instrument was expressly designed. The special throttle valves, by which the operator regulated the flow to keep a constant evaporation temperature, have proved amazingly delicate, and have enabled the operator to maintain a control which was practically perfect. This technique, therefore, made it possible to run the evaporation at the initial calorimeter temperature so that no corrections were necessary for changes in evaporation temperature.

A change in the distribution of liquid and vapor in the calorimeter occurs in all three types of experiment. This change is analyzed in the theory of the method, and is accounted for by use of the quantity

which is designated as beta. This quantity was determined calorimetrically throughout the range above 200°C .

Special details of experimental procedure which apply to the particular types of calorimetric measurements will next be described. These measurements were performed in a manner similar to those described in previous reports, save for differences peculiar to the apparatus or to the experimental range.

4. DESCRIPTION OF ALPHA EXPERIMENTS

It will be recalled that these experiments yield values of the change with temperature of the quantity, $H - Lu/(u' - u)$, denoted by the symbol α . Combined algebraically with values of $Lu/(u' - u)$ obtained from independent measurements, the data suffice to determine the change in enthalpy of the saturated liquid, H .

The procedure in these alpha experiments consisted essentially in heating a sample of water, part liquid and part vapor, over an accurately measured temperature range, and accounting for the energy added in the process. Two such heat-capacity determinations, a gross and a tare, are required for an evaluation of the change in α . These two determinations over the same temperature interval are made with different charges of water always maintaining the saturation condition. The one made with a large charge yields a gross value of energy added. The other, made with a small charge, yields a tare value. The difference or net value of energy added, divided by the net mass gives the value of the change in α for that temperature interval. This method of differences avoids the necessity of determining the heat capacity of the empty calorimeter.

The actual experiments were not limited to a single pair of measurements, but for each temperature interval, a group of measurements was made with several different charges of water, including both large and small charges. Two series of such groups were made, one in 10-degree intervals between 100 and 370°C , and another in 50-degree intervals between 100 and 350°C .

During the alpha experiments, the calorimeter was closed from connection with the upper tube by a metal diaphragm in the union at the top of the calorimeter, to avoid possible error from condensation of water in the upper tube.

Certain peculiarities characteristic of the behavior of the water sample in attaining thermal equilibrium at the higher temperatures have been noted in the previous report on vapor-pressure measurements made in this same apparatus. On account of this behavior there is a decrease in precision in accounting for the state as the critical region is approached. The results are also less precise in this region on account of the decrease in the net mass of water available when keeping the necessary saturation conditions. Greater tolerances are therefore unavoidable in the extreme upper range.

5. DESCRIPTION OF GAMMA EXPERIMENTS

These experiments yield values at a given temperature of the quantity, $L + Lu/(u' - u)$, denoted by the symbol γ . Combined algebraically with corresponding values of β , i. e., $Lu/(u' - u)$, the data suffice to determine the heat of vaporization, and thus complete the evaluation of the enthalpy, H' , of saturated vapor.

The procedure in these gamma experiments consists essentially in removing a definite mass of saturated vapor from the calorimeter at a constant temperature, and accounting for the energy added in this process. The disk which closed off the calorimeter at the upper union in the alpha experiments was punctured to allow passage of steam out the top.

Two different procedures were used in separate series of determinations, as an additional check on possible causes of systematic error. In one series, designated as method *A*, a number of successive determinations at a given temperature from a single charge of water were conducted as a continuous experiment. The evaporation and withdrawal of vapor were conducted as a steady process without interruption, but divided into separate determinations by directing the flow of vapor successively into different receivers. In this method the initial and final conditions in the calorimeter for each determination were evaluated with the process taking place at a steady state but not in a state of equilibrium.

In the other series, designated as method *B*, each determination was conducted as a completely separate experiment, starting and stopping with equilibrium prevailing in the calorimeter. The measurements made prior to March 23, 1934, were by method *A*, those after that date by method *B*. Each method has its advantages and disadvantages, but the results, where they could be compared with each other and with the results of the earlier measurements made with a different calorimeter, did not show any significant systematic difference as a result of either different manipulation or different apparatus.

Considerable investigation was made at high temperatures of the effect of the rate of withdrawal on the results of the experiments. It is reasoned that if liquid drops were entrained with the steam as a result of active boiling, the degree of wetness would be expected to increase with the rate of evaporation and with approach to the critical region. For this reason the rate of withdrawal was varied over a wide range in different experiments, especially at the higher temperatures where the effect should be greatest. No significant change in the results was obtained except near the critical temperature.

6. DESCRIPTION OF BETA EXPERIMENTS

To complete the determinations of heat content, H , from the results of the alpha experiments, and of heat of vaporization, L , from the gamma experiments, it is necessary to know the values of the quantity, $Lu/(u'-u)$, denoted by the symbol β . This quantity, which enters into the reduction as an insignificant correction term at 0° C, increases in magnitude with rising temperature, until at the critical state its limiting value becomes equal to that of gamma, when the heat of vaporization becomes zero.

It is shown in the theory [1] that beta may be determined calorimetrically as the amount of energy required per gram of saturated liquid withdrawn to just balance the heat expenditure in the evaporation which takes place in this process. Beta may also be determined by calculation, using the specific volume of the saturated liquid and the derivative of the vapor pressure in the formula, $\beta = T u d p / d t$. In fact, now that the specific volumes of liquid water have been determined

with such precision by Smith and Keyes [8] at Massachusetts Institute of Technology, it is only in the range of higher temperatures that the calorimetric method is comparable in precision with the direct volumetric measurements. As a part of the complete calorimetric program the measurements of beta were made and values obtained in the range from 100 to 374° C. These measurements were found to be confirmatory of the MIT smoothed-volume data, with the exception of small departures above 330° C. These beta measurements, together with dp/dt , constitute an independent evaluation of liquid specific volume, as well as a factor in evaluating the enthalpy.

The determination of beta was similar to a determination of gamma by method *B*, starting the flow and completing the experiment with the state of equilibrium prevailing in both cases. The essential difference was that liquid was withdrawn from the bottom of the calorimeter instead of vapor from the top. Regulation of the flow by a sensitive throttle valve was guided by the pressure indicator as before, to maintain constant evaporation temperature in the calorimeter. With careful attention given to the regulation of this valve, particularly at the beginning and end of the flow, it was possible to avoid any disturbance of the desired steady conditions sufficient to cause significant error.

Since it might be supposed that steam bubbles could be entrained in the withdrawn liquid, especially when near the critical region, tests were made of the effect of rate of withdrawal on the results in the same way as was done in the gamma experiments. Here again, no positive effect was observed which could be attributed to rate.

V. RESULTS OF MEASUREMENTS

The principal data from the three types of calorimetric experiments are given in tables 1, 2, and 3. (See pages 432, 437, 439.) These tables contain only the data remaining after sifting the record and discarding all experiments which were of doubtful worth on account of defects in the experimental technique. Examples of such defects were cases where fluid leaks occurred, where mass determinations failed to check properly, or where heat-leak control was faulty or the record deficient.

All known corrections for instrument calibrations and preliminary reduction of actual observed quantities have been made prior to these tabulations, which omit details of the complicated and bulky record of original observations. Further reduction and formulation of the results are next described.

1. REDUCTION AND FORMULATION OF DATA FOR ALPHA

From the alpha measurements, a group of data was obtained for determining the energy used in heating the calorimeter and its charge of water in each individual experiment. These data, corrected for instrument calibrations and for the several heat leaks, previously mentioned, are listed in table 1 as "Entire energy added." The four heat-leak corrections are combined and listed as "Heat leak." Since the values of entire energy are those for the actual measured increments of temperature, and since these differ slightly from the even values intended, the energy values were next adjusted by interpolation to correspond with the even temperatures at the 10-degree points. The values of $[Q]_1^2$ thus obtained were then ready for reduc-

tion to determine the change in alpha with temperature, $[\alpha]_1^2$, as a step toward deducing the change in enthalpy. The method of making the further reductions by deducing a calorimeter calibration from the experimental alpha data will next be described by starting with the general equation for constant mass or alpha experiments.

Taking account of the liquid in the lower tube below the calorimeter, the equation may be written

$$[Q]_1^2 = \left[E_A - VP + V \frac{L}{u' - u} + (M - m)\alpha \right]_1^2$$

in which $[Q]_1^2$ denotes the energy added to the calorimeter and its contents in heating through the even temperature interval, t_1 to t_2 , in a single experiment; M is the total mass of the charge introduced into the calorimeter system; m is the mass of the small portion (less than 0.5 percent) of the total charge in the connecting line outside the calorimeter boundary; E_A is the internal energy of the calorimeter; V is the internal volume of the calorimeter; P is the internal pressure; L is the heat of vaporization; and u' and u are specific volumes of saturated vapor and liquid, respectively; α is the specific property of saturated liquid water denoted by $H - \frac{Lu}{u' - u}$, where H is enthalpy of saturated liquid.

In this expression, the increments of temperature and the amounts of the charge are the independent variables, the resulting values of $[Q]_1^2$ are determined, as described above, from the observations.

Letting

$$\left[E_A - VP + V \frac{L}{u' - u} - m\alpha \right]_1^2 = [Z]_1^2,$$

the values of $[Z]_1^2$ or ΔZ are found from the sets of observations for each temperature interval by simple least-square solutions of the resulting equations of the form

$$[Z]_1^2 = [Q]_1^2 - M[\alpha]_1^2$$

The values of ΔZ from the 10-degree intervals were next adjusted by smoothing them with respect to temperature.

(This adjustment of the ΔZ 's was facilitated by separating each value into two parts. By rearranging, the above equation may be written

$$[Z]_1^2 = [E_A - m\alpha]_1^2 + \left[V \left(\frac{Tdp}{dt} - P \right) \right]_1^2$$

In this form, the last term contains quantities which are known with considerable accuracy and are relatively free from irregularities. This term was computed separately for each set of data and the values subtracted from the ΔZ 's. The part which remained was found to vary by only about 10 percent and almost linearly over the entire range of temperature. Since the heat capacity of the empty calorimeter makes up over 99 percent of this term, and since the material

of the calorimeter is known to have no abrupt transformations in the temperature range used, it is proper to adjust the experimental values by smoothing. A simple formulation of these values smoothed out accidental irregularities. The corresponding values of $\left[V\left(\frac{Tdp}{dt} - P\right)\right]_1^2$ were then restored to these smoothed values to give the smoothed values of ΔZ for each temperature interval. By this method of reduction and smoothing, there is deduced a calibration of the calorimeter as actually used in accordance with the theory. This calibration is analogous to the usual calorimeter calibrations for heat capacity, differing, however, in that the function ΔZ includes more than the mere heat capacity of the empty calorimeter.) These smoothed values of ΔZ are given as $[Z]_1^2$ for each 10-degree interval in table 1.

Using the adjusted values of ΔZ , each experiment now becomes a measurement of $\Delta\alpha$. The general equation may now be written

$$\Delta\alpha = [\alpha]_1^2 = \frac{[Q]_1^2 - [Z]_1^2}{M}$$

The values of $\Delta\alpha$ calculated thus are listed in table 1. It is evident that although experiments with both large and small charges are necessary in order best to evaluate ΔZ , the values of $\Delta\alpha$ from these low fillings are, in general, inferior in precision to those from larger masses. This is taken into account in calculating the mean values of $\Delta\alpha$.

The data from the 50-degree intervals, included in table 1, were made in two groups, designated as *I* and *II*, both of which were made earlier and with less perfect experimental technique than the measurements in 10-degree intervals. The summed heat-leak corrections were not applied to group *I* because of known causes of error in their evaluation. These errors would tend to balance out in the determination of $\Delta\alpha$, but not in the determination of ΔZ . Furthermore, some minor repairs made on the calorimeter after group *II* probably made small changes in the heat capacity and, therefore, in the ΔZ . For these reasons the measurements of groups *I* and *II* were calculated directly without using the adjustment of ΔZ applied to the 10-degree intervals.

The results of the determinations for the two 50-degree groups were then combined with assigned weights to be included with the data from the 10-degree intervals. In the final assembly and formulation of α , groups *I* and *II* do not greatly affect the final result. They do, however, comprise an independent though somewhat inferior series of measurements, which may be compared with the accepted formulation.

In addition to the measurements made with the present calorimeter and described above, other data were available in the interval of 100 to 270° C from observations made with the previous calorimeter and published in 1930. Since these independent data were in the same range as the later ones, they also were included in the revised and extended formulation made possible by the present new group of results. This formulation of the entire group of data for α will next be described.

The empirical equation deduced to represent smoothed and adjusted values of the increment of α , $[\alpha]'_{100}$, from the 100° point to any temperature, t , may be written

$$[\alpha]'_{100} = A(t-100) - B(t^2-100^2) - C \int_{100}^t 10^{-Dt^2} dt$$

where

$$\begin{array}{ll} A = 4.349321 & B = 4.23834 \times 10^{-4} \\ C = 0.153528 & D = 0.208103 \times 10^{-4} \end{array}$$

α is in int. j/g and t is in degrees centigrade. Values of α calculated by this equation are given in table 5 (see page 444), taking the value of $[\alpha]_{100}$ as 417.36, as previously determined in this laboratory [2].

Figure 8 shows graphically the values of the derivative $d\alpha/dt$, calculated by means of the above equation and plotted with respect to temperature. For comparison the values from the 1930 formulation, and the values as derived directly from the adjusted experimental data are shown. Figure 9 shows a graphical comparison between the values of $[\alpha]'_{100}$ from the different groups of measurements and the accepted formulation. This shows the accord of the separate series of measurements made at different times with different technique or different apparatus.

2. REDUCTION OF DATA FOR BETA AND GAMMA

The β and γ experiments were very similar in principle and with few exceptions, the reduction of the data was the same for both. The reductions for β will be first described.

Data were obtained from the β experiments for determining the energy used in the process of withdrawing saturated liquid from the calorimeter at a steady state. The measured electric energy added to the calorimeter was first corrected for electric-instrument calibrations and then for the several heat leaks previously mentioned. Although the corrections for residual heat leak were small, they were applied because systematic. Still another correction was applied to account for the small difference of the final temperature of an experiment from the initial temperature. This correction is for the energy which would bring the calorimeter and its contents, M_2 , at the end of an experiment, back to the initial temperature. The technique of the experiments was such that the change in temperature, Δt , was usually small. The corrections, ΔQ , were calculated by use of the equation

$$\Delta Q = \left(\frac{dZ}{dt} + M_2 \frac{d\alpha}{dt} \right) \Delta t,$$

taking the values of dZ/dt and $d\alpha/dt$ as determined in the alpha measurements. In a few of the later experiments, observations and corrections were made for a further small amount of energy caused by the transfer of a few milligrams of water between the calorimeter and the differential-pressure indicator. Since this correction is proportional to the mass so transferred and to the value of β , it is significant only when β is large. The values of these combined corrections to the entire energy added are listed as "Energy Corrections" in table 2. After applying these corrections to the observed data, the corrected values are listed as "Entire Energy Added."

The entire energy added was then divided by the mass of fluid withdrawn to give the energy added per gram, which is approximately equal to β . In most of the β experiments, it was necessary to correct this approximate value for the superheat of the liquid withdrawn.

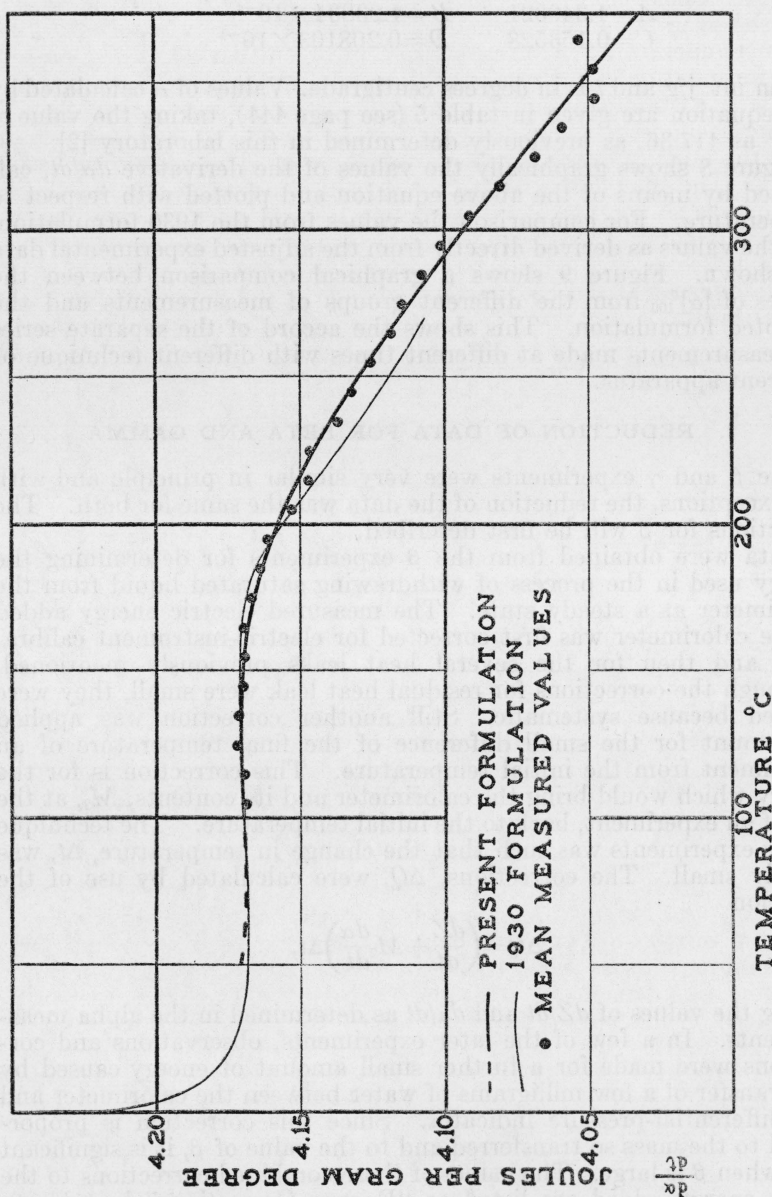


FIGURE 8.—Graph of the alpha derivative, $d\alpha/dt$.

This correction was computed from the observed temperature of the withdrawn liquid and the specific heat at constant pressure. It varied with the temperature and with the rate of withdrawal of liquid. It

was about 1 j/g at 200° C but was as large as 10 j/g at one of the higher temperatures.

Another correction was needed to reduce the individual values of β to common temperatures so that they may be combined and compared. Since all the experiments were conducted very close to a chosen even temperature, the individual values were reduced to these even temperatures by interpolation, final adjustments being made as the formulation of the results progressed. These corrections described above are combined and listed in table 2 as "Corrections to Beta."

For the most part, the reduction of the data from the γ experiments was the same as in the above description for reduction of observations for β . The chief exception is that since no detectable superheat of the vapor was found in the γ experiments, this correction was not required. The data for the γ experiments and steps in the reduction are given in table 3.

Having calculated the values of β and γ for each of the accepted experiments, a study was made of their deviations from the averages

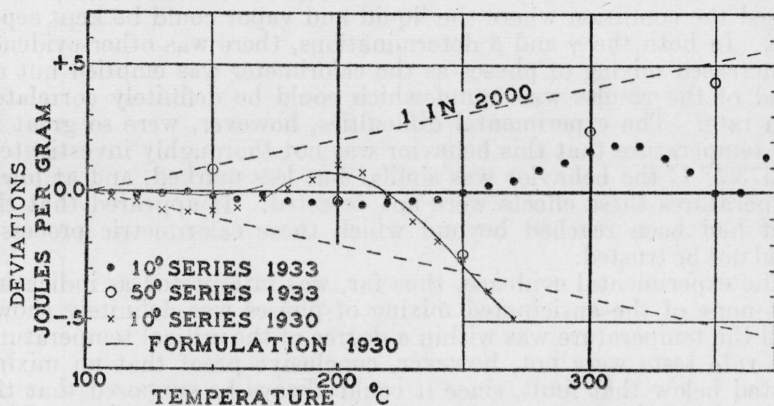


FIGURE 9.—Deviation of $\alpha_t - \alpha_{100}$ from present formulation.

for evidence of possible errors from mixing of phases and also to guide in weighting the results for the purpose of formulation.

One of the most commonly expressed causes for doubt of the reliability of data for heat of evaporation in the past has been the possibility of mixture of water drops in the steam. In the previous series of measurements in this laboratory, several measures had been taken to avoid this danger and to test its presence. Evaporation was then produced on the surface of a thin layer of water flowing over a heater of large area to promote quiet evaporation and thus avoid formation of any drops. Baffles were provided to intercept drops which might have been formed in spite of the first precaution. Finally, measurements were made at different rates of evaporation on the presumption that if drops were formed, the amount of liquid so entrained and carried out with the steam would vary with the rate and show a difference in the measured heat. No evidence of such an effect was found in the previous series of measurements extending from 50 to 270° C.

The present calorimeter was designed for higher working pressures and lacked a circulating pump. As a means of partially compensating for the lack of mechanical circulation, the evaporating process was

carried to the other extreme of actually inducing active boiling in the liquid. Baffles to intercept drops were, however, installed as before. The effect of mixing of phases was again looked for, even more carefully than before, and especially at the higher temperatures. It was reasoned that if there were mixing of phases, it would ordinarily be less, the lower the rate, and the results would consequently approach the normal value were it not for the lesser precision inherent in the lower rates. On the other hand, it seems certain that at some condition on approaching the critical state, the mixing of phases in these evaporation experiments would be unavoidable, regardless of the rate.

The experiments to investigate this effect were performed at widely different rates. In a number of cases at the higher temperatures, the rates ranged from 5 g/min down to less than 0.1 g/min. No definite indication of the effect was found in either the γ or β experiments even at temperatures up to and including 373° C.

At 374° C, however, the determinations of both γ and β were scattered much more widely than could be accounted for by inaccuracies of measurement. This scattering was attributed to having passed the condition where the liquid and vapor could be kept separate. In both the γ and β determinations, there was other evidence of increased mixing of phases as the calorimeter was emptied but no trend of the results was found which could be definitely correlated with rate. The experimental difficulties, however, were so great at this temperature that this behavior was not thoroughly investigated. At 373.5° C the behavior was similar but less marked, and at lower temperatures these effects were not detected. It appeared that the limit had been reached beyond which these calorimetric processes could not be trusted.

The experimental evidence, thus far, was interpreted as indicating that none of the anticipated mixing of phases was definitely shown until the temperature was within a degree of the critical temperature. The rate tests were not, however, conclusive proof that no mixing existed below that limit, since it could always be supposed that the rate was not changed enough to disclose the effect of mixing. Additional evidence by some entirely independent method would be more convincing, especially if obtained in the region well up toward the critical, for example, at 370° C or higher.

The desirability for such an additional check was further realized when the values of γ measured calorimetrically in this laboratory were compared with the values of specific volume of saturated vapor obtained by our colleagues in the steam research at the Massachusetts Institute of Technology, Messrs. Keyes, Smith, and Gerry [9]. Now that the vapor-pressure derivative is known with considerable accuracy, such comparisons are possible through the thermodynamic relations

$$\begin{aligned}\gamma &= T u' dp/dt \\ \beta &= T u dp/dt\end{aligned}$$

These comparisons, made as the two investigations progressed, indicated a satisfactory accord in the lower range of temperature but showed a progressively increasing discord in the upper range. These discrepancies were too large to be accounted for by experimental error in either laboratory. Since the deviation was in the direction which would be caused by wet steam in the calorimetric experiments,

these were placed under suspicion. On the other hand, the saturation values obtained by the MIT group were the result of extrapolations from measurements made on superheated vapor, and not on vapor actually in the saturated condition. A considerable gap between the saturation limit and the actual measurements was left to be bridged in this manner. It was appreciated that these extrapolations were less reliable the greater the interval to be covered, and the nearer the critical region was approached. Additional specific-volume measurements made closer to the saturation limit, would be evidence which might decide whether or not there was actually a defect in the calorimetric data on account of wet steam which the special tests had failed to disclose.

Such measurements were therefore undertaken, using the calorimeter as a volume-measuring instrument. The feasibility of using the calorimeter for specific-volume measurements depended upon the accuracy with which the volumetric capacity of the calorimeter at a chosen temperature and pressure could be determined. It may be recalled that this instrument was designed primarily for calorimetric measurements, but that it had also proved well suited for vapor-pressure measurements, nevertheless, by adoption of a suitable experimental technique, it was found possible to use this instrument with all the precision necessary to test the reliability of the calorimetric measurements. It was decided to make the volumetric measurements at a selected temperature in the region where doubt existed but not so close to the critical state as to preclude definite results.

3. CONFIRMATORY SPECIFIC-VOLUME MEASUREMENTS

The purpose of these measurements was not to establish the saturation limit of specific volume as a primary group of data but merely to secure additional evidence concerning the reliability of the calorimetric measurements. The temperature of 370° C was chosen as the test point after preliminary observations had shown it to be suitable for the purpose.

The measurements of the mass of the sample, its temperature, and its pressure were made in the same manner as in the calorimetric experiments. By use of the differential-pressure indicator, a series of determinations was made at one chosen pressure and at several temperatures in the vicinity of the saturated state. Measurements were made as close to the saturation limit as would allow the state of the sample to be definitely known. Other measurements were taken a little further away from the saturation limit in order to guide the extrapolation over the small interval to the limit. In using the differential-pressure indicator, the pressure in the air reservoir was first balanced against the chosen saturation pressure in the calorimeter and then used as a reference pressure for successive measurements made in either the superheated-vapor or compressed-liquid region at this pressure.

The capacity of the calorimeter at 370° C was determined by several calibration experiments. The use of the calorimeter as a volume-measuring instrument was complicated by the fact that it had a tube at the bottom, which extended beyond the thermally controlled envelope space out to a valve and pressure cell which were near the temperature of the room. This tube contained a small mass of liquid

which must be accounted for as a tare to be deducted from the gross fluid contents of the system. However, this tare mass was nearly constant even when the calorimeter temperature was raised up to 370°C . This was so because the emergent column of liquid remained at the room temperature, except for a short length between the tube heater and the water cooling cell just below it. The entire temperature gradient from room temperature up to the calorimeter temperature came in this section of tube, which was only 25 mm long and 1.7 mm in diameter (volume 0.057 cm^3). The temperature of all the calorimeter system above this gradient section could be controlled as in the calorimetric measurements, and hence could be filled with vapor while the tare mass of liquid filled the part below. The position of the boundary meniscus was very definite because it had to be where the saturation temperature comes in the region of large temperature gradient. The tare mass was determined with the net capacity filled with superheated vapor at 110°C at 1 atmosphere pressure. After deducting about 0.19 g for this vapor, the tare mass was found to be 1.33 g. This tare value was probably reliable within 0.01 g, regardless of the different mean density in the gradient portion when used at 370°C . This uncertainty was one of the least important of several.

The net capacity was determined at 300 and 340°C with the system filled with liquid at about 217 atmospheres pressure. At these temperatures the specific volume of the compressed liquid has been determined by Smith and Keyes [8], and their data have been taken as reference values for this calibration. The thermal expansivity of the material of the calorimeter shell was known from determinations made at this Bureau so that the capacity at 370°C could be calculated, giving, after a small additional correction for elasticity, the capacity at 207.77 atmospheres, corresponding to saturation pressure at 370°C .

The determinations at 300 and 340°C , thus corrected, gave values of 330.56 and 330.65 cm^3 , respectively, and a mean value of 330.6 cm^3 . A third measurement of capacity was made with the calorimeter filled with liquid at 42.2°C under 1 atmosphere pressure. The specific volume of water is more accurately known here, but the correction for temperature and pressure to reduce to 370°C was much larger, hence only small weight could be given to this experiment as a calibration. The reduced value of 330.97 cm^3 , which differed by about 1 part in 900 from the other measurements, was taken merely as a corroborative check.

The specific-volume measurements themselves were all made within 0.2° of 370°C . Seven measurements of the superheated vapor and six of the compressed liquid were made. In the vapor-volume experiments the reference air pressure in the pressure indicator was first balanced against the vapor pressure in the calorimeter at 370°C , with the calorimeter partly full of liquid. Next, the amount of fluid in the calorimeter was adjusted until, when allowed to reach a steady state with the reference pressure balanced, all parts of the calorimeter were slightly above the saturation temperature, as shown by readings of the thermoelements. This limiting condition was reached by making successive trials. When reached, it meant that the net capacity was filled with slightly superheated vapor, while the lower tube contained the tare mass of liquid.

The mean temperature of the calorimeter was then observed and the mass of fluid was determined in the usual way. This gave the

necessary data for determining a specific volume. Further withdrawal of a small amount of fluid and readjustment to a steady state at a slightly higher temperature gave data for another determination of the specific volume a little further out in the superheat.

Under the conditions of these experiments near 370°C , a gradient of two or three hundredths of a degree persisted in the calorimeter even after it reached a steady state. On this account, one determination made at a mean temperature only 0.01°C above saturation was distrusted because part of the net capacity was shown to be down to saturation temperature. The determination closest to saturation which could be trusted was at a mean temperature of about 0.03°C above saturation. Even at this point, there might have been question as to whether some of the inner surfaces of the calorimeter may not have still been wet on account of thermal lag. It might also be conceivable that liquid water could have been held in tiny capillary crevices, even though they were above the saturation temperature. Such a condition would of course give an erroneous result similar to the effect of wet steam in the calorimetric experiments on γ . To remove this uncertainty, a special test was made after one of the specific-volume determinations near 0.03°C superheat.

For this test the calorimeter was heated to about 2°C above the saturation temperature and allowed to remain an hour before cooling down again to the steady state at the reference pressure. The object of this heating was to evaporate any residual liquid and leave the surfaces dry. The calorimeter was then cooled by loss of heat from the outside, never allowing any part to go below the saturation temperature. If water had been evaporated by this process, the final temperature after cooling back to the original pressure should have been lower than before. Actually, no such change was found and it was therefore concluded that the determination could be trusted.

Five determinations were made between 0.03 and 0.06°C superheat and a sixth at about 0.16°C to give the trend of the isobar in this region.

The procedure for determining the specific volume of the compressed liquid was analogous to that for the superheated vapor except that the first measurements were made farthest from saturation. These measurements were made at temperatures between 0.16 and 0.17°C below the saturation temperature. The pressures were balanced against the pressure in the air reservoir kept over from the previous day's saturation balance. Some liquid was then withdrawn and two measurements made at about 0.05°C below saturation temperature. Further withdrawal brought the sixth measurement within 0.01°C of the saturation temperature. After this measurement, considerable water was withdrawn to establish saturation conditions again in the calorimeter to check the balancing air pressure in the air reservoir.

Very small departures from the intended constant pressure were observed in all the specific volume measurements. At the most, these corresponded to only 0.025 atmosphere on saturation pressure, or only 0.01° on saturation temperature. Where appreciable, corrections were applied to reduce the measurements to the pressure of 207.77 atmospheres corresponding to saturation at 370°C . The results of the specific volume measurements, thus reduced to this isobar, are given in table 4.

TABLE 4.—Value of specific volume along 207.77 atmospheres (370° C sat.) isobar

Temperature	Specific volume of liquid	Temperature	Specific volume of vapor
°C	cm ³ /g	°C	cm ³ /g
369.832	2.195	370.058	4.971
.836	2.195	.032	4.950
.835	2.195	.037	4.951
.950	2.213	.042	4.954
.956	2.213	.011	• 4.928
.994	• 2.230	.032	4.947
		.160	5.037

• Experiments under suspicion because of closeness to saturation.

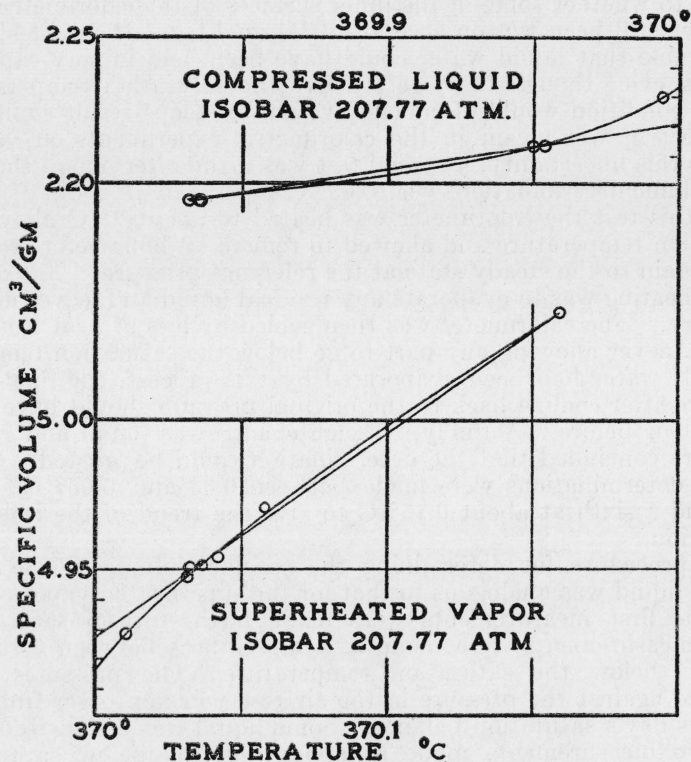


FIGURE 10.—Graphs of volumetric measurements.

The values from the experiments designated as suspicious may not have been erroneous but are perhaps not to be so fully trusted as the others because of the possibility that both phases may have been present in the volumetric space.

These results of the volumetric check measurements may now be examined for evidence bearing on the reliability of the calorimetric measurements, particularly in regard to the question as to whether or not the phases had been separated or partly mixed. Two graphs on figure 10 show how the observed isobaric volumes may be extrapolated to obtain values of specific volume at the 370° saturation limit. In each case, two suggested lines have been drawn, one a straight line,

disregarding the dubious near-saturation point, and the other line curved to include it. The increasing curvature in the latter case is in the sense expected from the known trend further away from the saturation limit, but it is also in the sense which would result from presence of both phases in the volumetric space.

In the case of the liquid, since the dubious point was approached from higher pressure it seems probable that no vapor space would have been formed even though the temperature at some point in the calorimeter may have been slightly above the saturation temperature. With this in mind, it seems fair to estimate the position of the experimental isobar at the saturation limit of 370°C as midway between the two drawn, thus giving a value of $2.227\text{ cm}^3/\text{g}$ for the specific volume of saturated liquid. A tolerance of $0.01\text{ cm}^3/\text{g}$ would seem to be adequate to cover both experimental error and uncertainty of this extrapolation. The mean value of β as derived from the reduced calorimetric observations is 359.4 int. j/g , while the adjusted value obtained from the formulation to be described later was 360.4 int. j/g . The specific volumes calculated from these values of β by the Clapeyron factor, Tdp/dt , are 2.220 and $2.226\text{ cm}^3/\text{g}$, respectively. The agreement between the two methods is within the above tolerance and, consequently, the calorimetric result is corroborated by the volumetric.

Now returning to the graph of the volumetric data for the vapor, there is perhaps greater likelihood that the value closest to saturation was too low on account of condensed liquid because parts of the container were below the saturation temperature. With this in mind, the value of $4.925\text{ cm}^3/\text{g}$ has been estimated from the volumetric data as the specific volume of saturated vapor at 370°C . The mean observed value for γ is 800.4 int. j/g , while the adjusted value obtained from the formulation to be described later is 798.7 int. j/g . The specific volumes calculated from these values of γ by the Clapeyron factor, Tdp/dt , are $4.944\text{ cm}^3/\text{g}$ and $4.933\text{ cm}^3/\text{g}$, respectively.

The mean observed calorimetric value is $0.019\text{ cm}^3/\text{g}$ higher than the preferred volumetric value. The formulated calorimetric value is 0.008 higher than the preferred volumetric value. These deviations are within a reasonable allowance for experimental errors. Moreover, the difference is in the opposite sense to the effect which would be caused by water mixed with the steam in the calorimetric measurements.

This experimental corroboration of both the γ and β measurements by independent volumetric measurements confirms the previous conclusion that these calorimetric results at 370°C show no decisive evidence of error attributable to mixture of phases, and that the increased spread of the results is the natural consequence of other inherent accidental errors.

This verification would perhaps be still more satisfactory if the deviations between the NBS and the MIT saturation values could be plausibly accounted for. Keyes, Smith, and Gerry, in their report [9] offer an explanation which, on examination, leads to a credible explanation and interpretation of these differences. On page 339 they suggest that their extrapolation may not be reliable, especially on approach to the critical region, and they give preference to other values for specific volume of saturated vapor in the temperature range above 350°C . The further examination of this matter can be

better made following the formulation of the calorimetric data, and it will therefore be resumed further on, where the formulated calorimetric data are available for comparison.

4. FORMULATION OF BETA AND GAMMA

From the study of the deviations of the individual values of β and γ from the averages, it was concluded to assign weights to these values in taking means at the even temperatures in preference to simple averages. For these means, weights were chosen proportional to the rate of withdrawal up to 2 g/min, above which rate the weight was taken as constant. This simple rule was in accord with judgment based on the physical facts and with estimates using the theory of probability. The mean values of β and γ at each even temperature are given in tables 2 and 3.

Deviations from these means were then used in estimating the weights to be assigned to the mean values in the formulation of the entire series with respect to temperature. Above 350° C, the assigned weights diminished rapidly with increasing temperature until at 374° C they were nearly zero. This is of course, the result of the behavior of the fluid and the limitation of the method on approach to the critical region. In other words, it means that even though the confirmatory tests at 370° C may have shown no gross errors to exist, there still is increasing uncertainty about determining the course of the saturation limit as the critical region is more closely approached.

In addition to the values of β and γ derived from this present series of measurements, there were other data available from measurements made with a previous calorimeter and published in 1930. These data include measurements of γ and β in the temperature range from 100 to 270° C, thus overlapping the range of the present determinations by 170° C. There were also the measurements of specific volume of liquid water by Smith and Keyes from which values of β could be calculated independently by use of the vapor pressure derivative, dP/dt . Values of β thus derived were used in the formulation in the range from 100 to 330° C. The values of β from calorimetric measurements in this range were in accord with those calculated from the volume data, but were less precise, and since the latter were from already smoothed and formulated data, they were the preferred choice. Above 330° C, the values of β from the calorimetric measurements were used since in this region there was an appreciable systematic difference from the volumetric values. A discussion of this choice of data on the saturated liquid may be found in the later section of this paper devoted to compilation of properties of saturated steam (page 424).

Having combined all the selected experimental values of γ and β to give single weighted mean values at certain even temperatures, empirical equations were deduced to represent smoothed and adjusted values. After considerable study of the fitting of the experimental values of γ and β to empirical equations, simultaneous equations for the sum and difference of these values were used in the formulation. From the theory, we have the relation $L = \gamma - \beta$, where L is the latent heat of vaporization and is one of the thermodynamic properties of steam sought. If we let $2D = \gamma + \beta$, D is the mean of γ and β , and in plane coordinates is the mean diameter of the figure formed by plotting γ and β against temperature. This mean diameter has less

curvature than γ or β and offers a means of extrapolation to a value at the critical temperature that is less speculative than the extrapolation of γ and β alone.

An equation for L was fitted to the weighted experimental mean values of γ and β in the range from 100 to 374.15° C. Another equation for D was fitted in the range from 330 to 374.15° C. The equation for D has the form

$$D = A + Bx^{0.617},$$

where $A = 535.08$; $B = 18.413$; $x = 374.15 - t$; and t is temperature in degrees centigrade. D is in int. j/g.

The equation for L has the form

$$L = a \left(\frac{374.15 - t}{100} \right)^{0.404} - b \left(\frac{310 - t}{100} \right)^{1.73} + c \left(\frac{165 - t}{100} \right)^{2.2},$$

where $a = 1585.19$; $b = 36.75304$; $c = 17.9218$; L is in int. j/g; t is in degrees centigrade; and $L = \gamma - \beta$.

The form of this equation has been chosen to conform with the accepted conventional interpretation that the latent heat approaches a zero value at the critical temperature at an infinite rate. As a result of this consideration, the temperature which is approached as a limit for real values of L is the numerical constant, 374.15, in the first term. This constant was determined to be the value which seemed to give the best fit with all the weighted experimental data. Although this evaluation of the critical temperature may not be decisive, yet it is an estimate which may be considered along with other recent published values.

Resuming the account of the formulation, after calculating from the equation for L , values in the range 100 to 374° C, and also values of D in the ranges 330 to 374° C, values of both γ and β could be found in the latter range by solution of the simultaneous equations. In the range from 100 to 330° C where the values of β calculated from specific-volume data of MIT had been adopted, values of γ were obtained by adding values of β to the corresponding values of L . The values of L , D , β , γ , and α , as finally deduced from the formulation of the experimental data, are given in table 5. The value of D yielded by the formula at the temperature of 374.15° C is the identical value of γ and β at this point where $L = 0$.

VI. COMPILATION OF PROPERTIES OF SATURATED STEAM

Under this head the adjusted experimental data are assembled, combined, and tabulated in terms of the specific properties usually compiled in steam tables. Along with the description of this compilation, certain groups of similar data from other research laboratories have been examined, using graphs to exhibit the degree of accord. In this examination the aim has been to note considerations which apply to relative appraisals, rather than to attempt a general appraisal. In most cases, where differences were found, it has been possible to reconcile them as the result of interpretation of measurements, rather than to attribute them to experimental errors.

It has been shown above how adjusted and smoothed values of the calorimetric data have been deduced through use of empirical formulas. These computed values of α , β , and γ , as given in table 5, define the behavior of steam as far as the calorimetric survey goes. From them, corresponding values of enthalpy and entropy may be derived. By use of the vapor-pressure derivative, specific volumes also may be derived. Values of the factor $T dp/dT$ have been given in table 5, using the derivative dp/dT from Osborne and Meyers' [10] compilation.

Enthalpy of saturated liquid is derived from α and β according to the relation

$$H_t - H_0 = \alpha_t - \alpha_0 + \beta_t - \beta_0$$

The value of β_0 is 0.012 int. j/g.

Enthalpy of saturated vapor is derived from α and γ according to the relation

$$H'_t - H_0 = \alpha_t - \alpha_0 + \gamma_t - \beta_0$$

Entropy, ϕ , of saturated liquid is derived according to the relation

$$\phi \Big|_0^T = \int_0^T \frac{d\alpha}{TdT} dT + \frac{\beta}{T} \Big|_0^T$$

This formula may also be written

$$\phi \Big|_0^T = \frac{H}{T} \Big|_0^T + \int_0^T \frac{\alpha}{T^2} dT$$

As a check, the numerical calculations have been made by means of both forms.

The value of the entropy of saturated liquid at 100° C, 0.3120 IT cal⁴ per gram degree, has been recalculated from the data in the report published in 1930 [2], using 273.16 as the absolute temperature of the ice point instead of 273.11, which value was used in that report.

Entropy, ϕ' , of saturated vapor is derived according to the relation

$$\phi' = \phi + L/T$$

Calculated values of entropy, specific volume, enthalpy, latent heat, and vapor pressure, expressed in the units used in the International Steam Table are given in table 6.

The numerical values for properties of steam given in tables 5 and 6 represent the appraisal and formulation of the calorimetric measurements now completed in the range above 100° C. They supersede previous compilations which have been published from time to time as progress of the work advanced.

The adjusted and smoothed values of the properties of saturated steam having now been deduced from the experimental data, it will be convenient to resume examination of similar data from other sources. The first examination to be so made is the comparison of the MIT values for specific volume of saturated vapor.

⁴ The International Steam Table calorie, denoted by the symbol IT cal, is defined as 1/860 international watt-hour which is equivalent to 4.18605 international joules. The report of the Third International Steam Table Conference held in America in September 1934, is given in *Mechanical Engineering* for November 1935.

This comparison is made on the basis of gamma ($Tu' dp/dt$), since the values of this property are deducible from the MIT data for volume and vapor pressure, and are better proportioned for graphical comparison than are the specific volumes directly. The values of dp/dt from the MIT report on vapor pressure [11] were used for this conversion, since these have been preferred by the MIT group for the similar calculation of latent heat by the Clapeyron equation. Values of gamma thus obtained from the calculated values in table VI (MIT) [9] compared to the formulated values of gamma from table 5 of this paper are shown in figure 11, plotted with reference to temperature. From table 5 (MIT), deviations of the graphically extrapolated saturation volumes from the smoothed values given by equation 10 (MIT), may be similarly converted to deviations of gamma by use of the

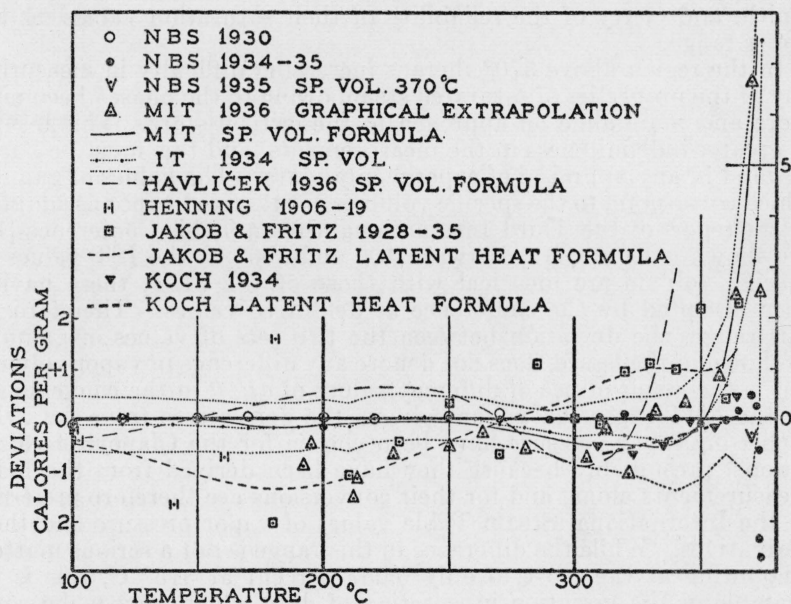


FIGURE 11.—Comparison graph for saturated vapor.

Clapeyron factor, $T dp/dt$. These deviations of the MIT extrapolated values from their smoothed values have also been plotted.

Inspection of the graph shows that in addition to the irregularities attributable to accidental errors of measurement, there appears to be a tendency to systematic departure of the MIT empirical formula 10 from the graphical extrapolation. Keyes, Smith, and Gerry have observed these deviations and have discussed the difficulty of graphical extrapolation, giving preference to saturation values deduced through their general formulation by an equation of state. A reason for the systematic tendency of the graphic extrapolation to low values at the lower temperatures is found in their statement that the effect of adsorption on the walls of the container tends to its largest effect near saturation and at the lower temperatures. They say elsewhere that "to give reliable saturation volumes, the analytical method using smoothing functions requires isometric pressures and temperatures

close to saturation." To reconcile these apparently conflicting views it may be inferred that an intrinsic characteristic of the equation of state has obviated the latter requirement at the lower temperatures. It is interesting to note that, in the range of temperature from 100 to 250° C, the MIT formulation is in much better accord with the NBS calorimetric values than with the volumetric measurements nearest to saturation. It may be that the closeness of this accord is partly accidental, and it would therefore be a mistake to assume that it is a measure of the experimental accuracy.

From 250 to 350° C the accord of the two independent formulations is not so good but is perhaps within a reasonable allowance for experimental error.

Above 350°, and especially at 370°, the experimental evidence given previously in this paper confirms the doubt expressed by Keyes, Smith, and Gerry of the reliability of their saturation values in this upper range.

In the region above 370° there is increasing difficulty in measuring any of the properties of saturated steam owing to the phases becoming more and more alike on approach to the critical state. This results in greater indefiniteness in the measurements, and this must be borne in mind in any appraisal of experimental data. The values of gamma which correspond to the specific volumes of saturated vapor as adopted in the report of the Third International Steam Table Conference [12] are shown in figure 11. Between 100 and 360° C, the IST values of specific volume are identical with those of the MIT, these having been accepted by the conference as definitive values. Therefore, in this range, the deviation between the two sets of values of gamma, as shown in the figure, does not denote any difference in vapor volume, but is the result of use of different values of dP/dt in the conversions. In the MIT report on vapor volume, the derivatives from the MIT report on vapor pressure have been chosen for the Clapeyron calculations, presumably because they have been derived from the MIT measurements alone, and for their conversions are therefore preferred to the International Steam Table values of vapor pressure and their derivatives. While the difference in this range is not a serious matter, amounting at the most to only 0.057 percent at 315° C, this is an example of the variation in selection of data which may make comparisons appear either more or less in accord.

Another example illustrating this point is found in the present paper, where in the choice of values of specific volume of saturated liquid water from 100 to 330° C to use in formulating, those of the MIT have been chosen in preference to the values from the NBS calorimetric measurements of beta. Although the latter measurements were in satisfactory agreement, it was recognized that the formulated MIT values were more precise in most of this range. The reason for restricting the choice to this range is explained further on, where limitation of the extrapolation is discussed (page 427).

Another group of saturation data is the series of latent-heat measurements made at the Reichsanstalt. This work was started by Henning over 30 years ago and has been since continued by Jakob and Fritz. The measurements have extended over the temperature range from 30 to 365° C and have involved several modifications of the original apparatus.

The assembled results of this extensive research are given by Jakob and Fritz [13] in their recent report. In this report an empirical formula for latent heat of vaporization is given and also values calculated therefrom, extending from 0 to 372° C. Corresponding values of specific volume of saturated vapor, computed by use of the Clapeyron equation, are also given. These experimenters used a method in which the measured quantity, called by them "spezifischer Wärmeaufwand", is the same as the gamma of the present paper. This quantity will therefore be taken as the basis of comparisons in preference to the values of latent heat which have been deduced from the experimental data by use of a correction which corresponds to the beta of this paper. The values of beta used by Jakob and Fritz were calculated from the Smith and Keyes data for specific volume of saturated liquid water, using the vapor-pressure derivative from the vapor-pressure tables of Osborne and Meyers. The values of gamma corresponding to the smoothed values of latent heat have been calculated by restoring these values of beta according to the relation

$$\gamma = L + \beta$$

These smoothed values of gamma from the Reichsanstalt data have been plotted as deviations from the NBS gamma taken as reference. Deviations of the experimental or measured values as given by Jakob and Fritz in their table 8 have also been plotted.

By inspection of this graph, an idea may be gained of the relative accord of the different independent series of calorimetric measurements with each other and with the respective formulations by which the experimental data are smoothed and interpreted.

The method of these experimenters employed the same general principle as that used at the National Bureau of Standards. In both laboratories, the water is evaporated in a calorimeter by measured electric heating, the vapor withdrawn from the calorimeter, condensed, and collected in a receiver for determination of the mass. There are, however, several features of the details of the apparatus, and of the technique of manipulation, which distinctly differentiate the two methods. Aside from the relative size and the degree of thermal insulation of calorimeters from envelopes, one other marked difference is the means of handling the withdrawn vapor and bringing it down from the saturation state in the calorimeter to the state in the receiver. The National Bureau of Standards technique was to throttle the vapor through a sensitive adjusting valve down to the low pressure before leading outside into a detachable receiver where it was condensed and weighed. The Reichsanstalt technique was to collect the condensate in a receiver where the total pressure was kept nearly equal to the pressure in the calorimeter by an artificial atmosphere of gas. In this scheme the pressure of the steam is reduced by diffusion through the gas to the condensation pressure. From the receiver it was transferred to a weighing vessel at atmospheric pressure. There were also considerable differences in the manner of observing temperatures and temperature distribution. It is not possible to estimate how these and other differences in technique may have affected the results.

It will be recalled that in the two regions where the deviation between the results of the two laboratories is most appreciable, the

NBS results have been doubly established. In the lower range from 100 to 270° C, this verification was by repetition of the measurements in an entirely different calorimeter, while at 370° C the determinations by calorimetric measurement were corroborated by volumetric measurements. The differences which remain between the final smoothed formulations of the results may perhaps be taken as evidence of the inherent difficulty of this type of physical measurement.

The latent heat in the pressure range from 100 to 200 kg/cm² (310 to 365° C) has been determined by We. Koch [14] at the Technische Hochschule München, using an entirely different method of calorimetry than either the NBS or the PTR. An electrically heated flow calorimeter was used. Compressed liquid water at a determined temperature and pressure entered the calorimeter, where on receiving the energy supplied by a measured electric-power input, the water was heated, evaporated, and the vapor superheated to a state determined by the observed temperature and pressure. Experimental data obtained from separate measurements on the heat capacity of compressed liquid water and on superheated vapor, when extrapolated to the saturation limit, furnished the corrections by which the latent heat could be deduced from the evaporation measurements. The water rate was determined by weighing the water condensed during a determined time while a steady rate was kept. The formula for latent heat given by Koch is similar in type to those given by Jakob and Fritz, and to the one given in this paper, but fewer terms are required in Koch's formula because of the lesser interval covered.

The smoothed values calculated by Koch ⁵ have been plotted in figure 11 as deviations from the NBS formula for latent heat, assuming that these deviations are the same for gamma as for latent heat, or that beta is the same for both. Koch's determinations of latent heat are subject to any uncertainty which there may be in the extrapolations to the saturation limit, in addition to any experimental errors of measurement. Nevertheless, the degree of accord of this determination by a dissimilar calorimetric method is noteworthy, particularly toward the higher temperatures, where it is in contrast to the deviations of the Jakob and Fritz values.

Still another evaluation of values of properties of saturated steam is found in the recent publication of Havlíček and Mískovský [15]. These authors present the results of an extensive research conducted at the Witkowitz Collieries in Moravska Ostrava, Czechoslovakia. They have measured the enthalpy of superheated steam in the range of pressures from 1 to 400 kg/cm² and in the range of temperatures from 20 to 550° C. The experimental method used a condensing calorimeter by which the heat given out by steam, initially at a determined temperature and pressure, is absorbed by cooling water and thus measured. A large number of points were thus established in the enthalpy field of superheat, including one isobar very near the critical pressure.

The experimental results of this calorimetric survey have been formulated by Juza [15], a coworker, in a supplementary contribution, using an equation of state somewhat analogous to that of the MIT. There is, however, this distinct difference in that, while the MIT group deduce enthalpies from their volumetric measurements,

⁵ An error in calculation at 245° has been corrected. The value is 229.9 instead of 230.4 cal/g.

Juza deduces specific volumes from the calorimetric measurements of enthalpy obtained by Havlíček and his associates.

From this formulation, when supplemented by vapor-pressure data, there is deduced a series of values of specific volume and enthalpy of saturated vapor, which may be compared with the results of the present paper. For this comparison, the specific volumes are reduced to the corresponding values of γ , as in the comparison of the MIT results. These Havlíček-Juza values of γ are plotted in figure 11 as deviations from the NBS values.

It should be borne in mind, in any appraisal of saturation values of either specific volume, latent heat, or vapor enthalpy that the saturation values from the Havlíček survey, and likewise those from the MIT measurements, as well as those of Koch, all depend on extrapolation of one kind or another from measurements at other than the saturation state. It should, therefore, be well considered whether these indirect evaluations and direct measurements upon the saturated fluid should be regarded as equally definitive. In this connection the completeness with which the path of approach to saturation is experimentally established may be fully as important a factor as accuracy of measurements.

As an extreme example of the uncertainty in extrapolating superheat data to the saturation limit, figure 12 shows two graphical extrapolations of a 225 kg/cm² isobar of enthalpy, along with the saturation limit of enthalpy from the NBS measurements. The superheat data were taken from Havlíček's measurements. It is apparent that without experimental points nearer to saturation than the 3.9° in the compressed liquid or 6.1° in the superheat the location of the saturation limit by means of extrapolation, may be uncertain by from 20 to 40 calories. Many other less extreme cases may be found in study of the experimental data which are now available.

One of these which has a direct bearing on the choice of auxiliary data used in formulating the results of the NBS measurements, is illustrated graphically in figure 13. This graph exhibits the experimental data for specific volume of liquid water from the work of Smith and Keyes [8]. Not all their results are shown, but only those in the region where certain differences are found between their derived saturation values and those of the NBS. The portions near saturation of six of their isothermals are shown plotted against pressures. The observed points are shown, also the calculated saturation values, and the NBS saturation values. For the 320 and 330° C isothermals, smooth curves fit all the points well, including the NBS values.

For the 340, 350, and 360° isothermals, there is increasing discrepancy between the calculated saturation values and the trend indicated by the experimental points, which is confirmed by the NBS values. At 350° the entire course of the calculated isothermal differs systematically from the experimental.

The 370° isothermal approaches the saturation limit so gradually that it would be impossible to determine the saturation limit from the trend beyond the nearest experimental point, 5.75 atmospheres away.

From a consideration of these facts it was concluded that while no conflict was shown between the values deduced from the NBS beta measured calorimetrically, and the actual measurements of the MIT on compressed liquid water, the saturation values deduced by Smith and Keyes from the MIT measurements above 330° were open to

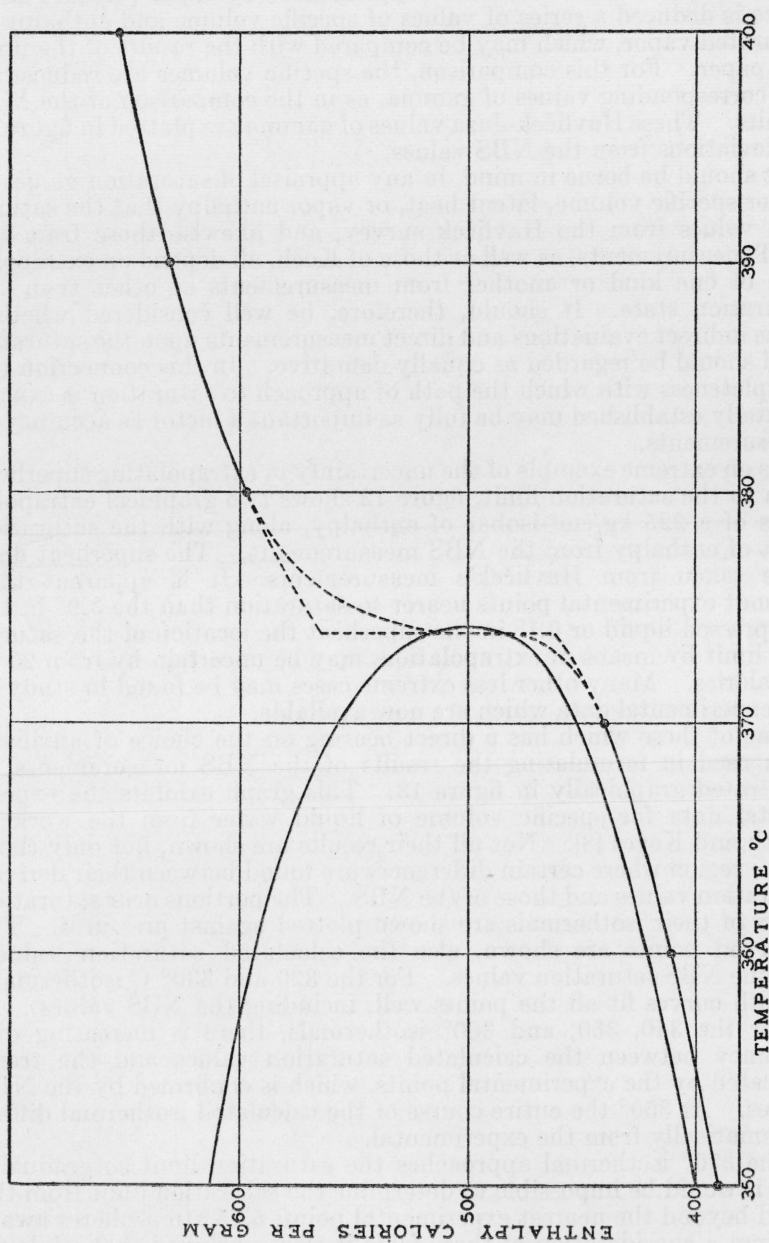


FIGURE 12.—Example of uncertain extrapolation.

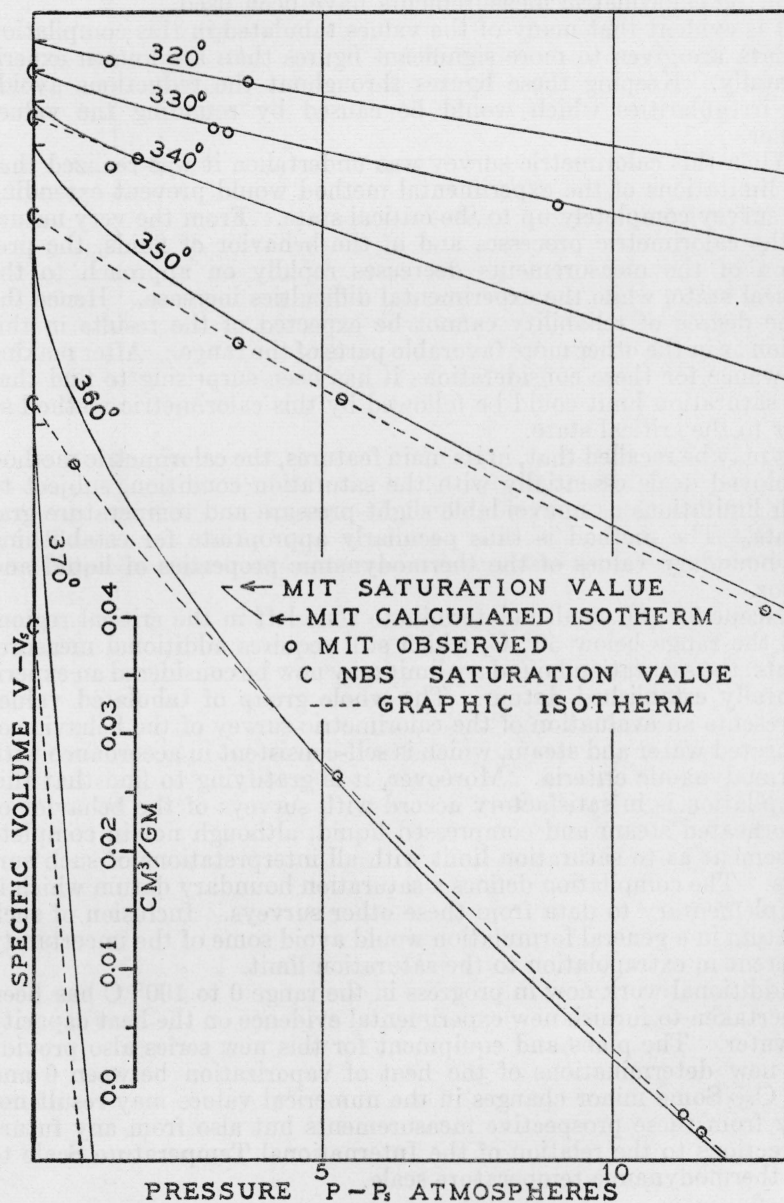


FIGURE 13.—Extrapolation of liquid specific volumes to saturation limit.

question. The NBS compilation has, therefore, accepted the values from the MIT formulation only up to and including the 330° point for the values of beta used, while above this point values formulated from the calorimetric measurements have been used.

It is evident that many of the values tabulated in this compilation of data are given to more significant figures than are known experimentally. Keeping these figures throughout the reductions avoids the irregularities which would be caused by rounding the values sooner.

When this calorimetric survey was undertaken it was realized that the limitations of the experimental method would prevent extending the survey completely up to the critical state. From the very nature of the calorimetric processes and of the behavior of fluids, the precision of the measurements decreases rapidly on approach to the critical state, while the experimental difficulties increase. Hence the same degree of reliability cannot be expected of the results in this region as in the other more favorable parts of the range. After making allowance for these considerations it has been surprising to find that the saturation limit could be followed by this calorimetric method so near to the critical state.

It may be recalled that, in its main features, the calorimetric method employed deals essentially with the saturation condition, subject to such limitations as unavoidable slight pressure and temperature gradients. The method is thus peculiarly appropriate for establishing the boundary values of the thermodynamic properties of liquid and vapor.

Except for the small indeterminate zone left in the critical region and the range below 50° C, which still requires additional measurements, the saturation boundary limit may now be considered an experimentally established datum. The whole group of tabulated values represents an evaluation of the calorimetric survey of the behavior of saturated water and steam, which is self-consistent in accordance with thermodynamic criteria. Moreover, it is gratifying to find that this compilation is in satisfactory accord with surveys of the behavior of superheated steam and compressed liquid, although not in complete agreement as to saturation limit with all interpretations of such surveys. The compilation defines a saturation boundary datum which is complementary to data from these other surveys. Inclusion of such a datum in a general formulation would avoid some of the uncertainty inherent in extrapolation to the saturation limit.

Additional work now in progress in the range 0 to 100° C has been undertaken to furnish new experimental evidence on the heat capacity of water. The plans and equipment for this new series also provide for new determinations of the heat of vaporization between 0 and 50° C. Some minor changes in the numerical values may result not only from these prospective measurements but also from any future corrections to the relation of the International Temperature Scale to the thermodynamic temperature scale.

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WASHINGTON, February 10, 1937.

VIII. TABLES OF DATA AND PROPERTIES OF STEAM

TABLE 1.—Principal data from a experiments

Date	Mass of water, <i>M</i>	Initial temperature	Final temperature	Heat leak	Entire energy added <i>Q</i>	Energy for even temperature interval Q_{11}^2	α_1 , ²	Deviation $\alpha_1 - \alpha_m$
	g	°C	°C	Int. j	Int. j	Int. j	Int. j/g	Int. j/g
6-6-33	248.316	100.024	110.122	-6.1	17,043.0	16,876.8	41.689	0.002
6-7-33	43.074	100.016	110.052	-0.1	8,351.6	8,320.6	41.691	.004
6-8-33	43.074	100.082	110.078	-9	8,317.1	8,320.2	41.682	-.005
6-12-33	250.163	100.052	110.096	-9	17,028.1	16,953.0	41.686	-.001
6-14-33	44.159	100.020	110.175	-6.6	8,495.6	8,365.1	41.674	-.013

Even-temperature interval t_1 to $t_2=100$ to 110° . $Z]_{100}^{110}=6,524.8$ int. j. Mean value of $\alpha]_{100}^{110}=41.687$ int. j/g.

6-5-33	248.316	109.993	119.900	-2.0	16,792.5	16,950.4	41.699	.002
6-6-33	248.316	110.122	120.132	-7.7	16,968.3	16,951.1	41.702	.005
6-7-33	43.074	110.052	120.198	-0.3	8,515.7	8,393.2	41.726	.029
6-8-33	43.074	110.078	119.987	-5	8,314.3	8,390.6	41.666	-.031
6-12-33	250.163	110.096	119.976	-6.2	16,822.7	17,026.4	41.695	-.002
6-14-33	44.159	110.175	119.798	-0.6	8,117.4	8,436.1	41.672	-.025

Even-temperature interval t_1 to $t_2=110$ to 120° . $Z]_{110}^{120}=6,595.9$ int. j. Mean value of $\alpha]_{110}^{120}=41.697$ int. j/g.

6-5-33	248.316	119.900	129.988	-7	17,178.8	17,029.8	41.713	-.001
6-6-33	248.316	120.132	130.118	-6.0	17,005.8	17,028.6	41.708	-.006
6-7-33	43.074	120.198	130.058	0.7	8,353.5	8,471.1	41.772	.058
6-8-33	43.074	119.987	130.032	-5	8,506.8	8,468.4	41.710	-.004
6-12-33	250.163	119.976	130.076	-6.1	17,277.3	17,107.3	41.715	.001
6-14-33	44.159	119.798	129.841	-0.5	8,548.4	8,513.5	41.706	-.008

Even-temperature interval t_1 to $t_2=120$ to 130° . $Z]_{120}^{130}=6,671.8$ int. j. Mean value of $\alpha]_{120}^{130}=41.714$ int. j/g.

6-5-33	248.316	129.988	139.963	-7	17,069.2	17,113.0	41.717	.002
6-6-33	248.316	130.118	140.055	-6.1	17,008.4	17,114.4	41.722	.007
6-7-33	43.074	130.058	140.088	-0.5	8,578.9	8,552.5	41.751	.036
6-8-33	43.074	130.032	140.109	-8	8,617.8	8,551.3	41.724	.009
6-12-33	250.163	130.076	140.136	-6.7	17,294.2	17,188.4	41.710	-.005
6-14-33	44.159	129.841	140.181	-0.5	8,886.9	8,593.5	41.654	-.061

Even-temperature interval t_1 to $t_2=130$ to 140° . $Z]_{130}^{140}=6,754.1$ int. j. Mean value of $\alpha]_{130}^{140}=41.715$ int. j/g.

6-5-33	248.316	139.963	150.216	-6	17,637.1	17,199.4	41.696	.010
6-6-33	248.316	140.055	150.093	-7.5	17,263.3	17,198.3	41.691	.005
6-7-33	43.074	140.088	150.040	-0.6	8,601.2	8,641.8	41.698	.012
6-8-33	43.074	140.109	150.081	-9	8,618.2	8,641.6	41.693	.007
6-12-33	250.163	140.136	150.123	-6.7	17,248.7	17,270.9	41.674	-.012
6-14-33	44.159	140.183	149.904	0.6	8,444.4	8,686.9	41.695	.009

Even-temperature interval t_1 to $t_2=140$ to 150° . $Z]_{140}^{150}=6,845.7$ int. j. Mean value of $\alpha]_{140}^{150}=41.686$ int. j/g.

6-5-33	248.316	150.216	160.190	-4	17,246.5	17,290.5	41.678	-.005
6-6-33	248.316	150.093	160.164	-6.6	17,420.8	17,296.0	41.700	.017
6-7-33	43.074	150.040	159.940	-0.2	8,649.0	8,736.6	41.682	-.001
6-8-33	43.074	150.081	159.946	-6	8,619.5	8,737.4	41.700	.017
6-12-33	250.163	150.123	160.018	-6.9	17,181.3	17,363.9	41.664	-.019
6-14-33	44.159	149.904	159.796	-0.3	8,686.5	8,782.7	41.702	.019

Even-temperature interval t_1 to $t_2=150$ to 160° . $Z]_{150}^{160}=6,941.2$ int. j. Mean value of $\alpha]_{150}^{160}=41.683$ int. j/g.

6-5-33	248.316	160.190	170.082	.4	17,215.0	17,401.9	41.705	.012
6-6-33	248.316	160.164	170.164	-6.7	17,398.0	17,396.6	41.684	-.009
6-7-33	43.074	159.940	169.924	-0.5	8,827.9	8,843.7	41.740	.047
6-8-33	43.074	159.946	169.877	.3	8,779.5	8,841.3	41.684	-.009
6-12-33	250.163	160.018	169.937	-6.8	17,333.4	17,474.4	41.687	-.006
6-14-33	44.159	159.796	170.103	-0.6	9,158.2	8,885.7	41.665	-.028

Even-temperature interval t_1 to $t_2=160$ to 170° . $Z]_{160}^{170}=7,045.8$ int. j. Mean value of $\alpha]_{160}^{170}=41.693$ int. j/g.

Even-temperature interval t_1 to $t_2=230$ to 240° . $Z|_{230}^{240}=8.008.9$ int. j. Mean value of $\alpha|_{230}^{240}=41.400$ int. j/g.

TABLE 1.—Principal data from a experiments—Continued

Date	Mass of water, M	Initial temperature	Final temperature	Heat leak	Entire energy added Q	Energy for even temperature interval Q_1	α_1	Deviation $\alpha_1 - \alpha_m$
	g	°C	°C	Int. j	Int. j	Int. j	Int. j/g	Int. j/g
5-29-33.....	67.996	330.275	340.149	-4.8	13,153.2	13,312.4	40.591	-0.001
5-31-33.....	67.996	329.890	340.306	0.3	13,871.9	13,313.1	40.601	.009
6-1-33.....	175.636	329.930	339.979	-1.6	17,760.8	17,674.9	40.553	-.039
7-21-33.....	161.675	330.043	339.958	-1.6	16,976.6	17,121.8	40.633	.041
Even-temperature interval t_1 to $t_2 = 330$ to 340° . $Z_{330}^{340} = 10,552.4$ int. j. Mean value of $\alpha_{330}^{340} = 40.592$ int. j/g.								
5-29-33.....	67.996	340.149	350.051	-1.6	13,638.8	13,768.0	40.257	-.222
5-31-33.....	67.996	340.306	350.143	0.1	13,565.4	13,777.2	40.392	-.087
6-1-33.....	175.636	339.979	350.319	-1.2	18,785.6	18,158.6	40.683	.104
7-21-33.....	161.675	339.958	349.959	-4.0	17,576.5	17,578.2	40.498	.019
Even-temperature interval t_1 to $t_2 = 340$ to 350° . $Z_{340}^{350} = 11,030.7$ int. j. Mean value of $\alpha_{340}^{350} = 40.479$ int. j/g.								
7-21-33.....	161.675	349.959	360.051	-3.0	18,391.9	18,224.0	40.495	.023
7-24-33.....	154.806	350.031	360.051	-2.7	17,973.4	17,933.6	40.416	-.056
7-25-33.....	83.182	350.024	360.168	-0.7	15,270.3	15,045.2	40.493	.021
7-26-33.....	140.531	350.085	360.037	-5.6	17,293.9	17,373.4	40.536	.064
7-27-33.....	140.044	349.944	360.067	-4.8	17,518.5	17,304.4	40.184	-.288
9-28-33.....	94.582	350.046	360.034	-0.6	15,471.6	15,487.6	40.290	-.182
10-5-33.....	81.038	350.032	360.301	-6.9	15,395.1	14,977.7	40.732	.260
10-6-33.....	81.557	350.023	360.244	-6.5	15,337.7	14,995.2	40.687	.215
10-9-33.....	139.949	350.038	360.294	-14.2	17,814.8	17,355.5	40.576	.104
Even-temperature interval t_1 to $t_2 = 350$ to 360° . $Z_{350}^{360} = 11,676.9$ int. j. Mean value of $\alpha_{350}^{360} = 40.472$ int. j/g.								
7-26-33.....	140.531	360.037	369.738	-33.6	17,878.3	18,449.2	40.652	.116
7-27-33.....	140.044	360.067	369.930	-23.7	18,139.9	18,393.3	40.394	-.142
9-28-33.....	94.582	360.034	369.821	-10.2	16,193.6	16,560.2	40.428	-.108
10-5-33.....	81.038	360.306	370.164	-39.8	15,814.2	16,011.1	40.409	-.127
10-6-33.....	81.557	360.244	369.978	-34.2	15,625.2	16,039.7	40.503	-.033
10-9-33.....	139.949	360.294	369.866	-19.1	17,654.4	18,436.4	40.729	.193
Even-temperature interval t_1 to $t_2 = 360$ to 370° . $Z_{360}^{370} = 12,736.4$ int. j. Mean value of $\alpha_{360}^{370} = 40.536$ int. j/g.								
1-17-33.....	45.854	99.960	150.153	-----	43,129.2	42,960.3	208.466	-.216
1-18-33.....	45.854	99.966	150.181	-----	43,162.2	42,973.8	208.760	.078
1-19-33.....	45.854	100.062	149.796	-----	42,724.7	42,956.0	208.372	-.310
1-21-33.....	218.231	100.196	149.730	-----	78,184.1	78,922.6	208.592	-.090
1-23-33.....	218.231	99.950	149.363	-----	77,977.9	78,920.0	208.580	-.102
1-24-33.....	218.231	100.003	149.743	-----	78,557.3	78,972.9	208.823	.141
1-25-33.....	218.231	99.984	150.168	-----	79,242.6	78,949.2	208.714	.032
1-26-33.....	61.155	99.978	150.149	-----	46,349.2	46,189.0	209.103	.421
1-27-33.....	61.155	100.052	150.241	-----	46,340.7	46,159.8	208.626	-.056
Group I. $Z_{100}^{150} = 33,401.3$ int. j. $\alpha_{100}^{150} = 208.682$ int. j/g = α_m .								
5-9-33.....	221.000	99.990	149.766	-5.4	79,123.9	79,484.8	208.542	-.025
5-10-33.....	125.970	100.070	150.037	-2.9	59,638.5	59,676.3	208.616	.049
5-11-33.....	59.989	100.105	150.143	-1.6	45,947.2	45,907.0	208.538	-.029
5-12-33.....	59.989	100.062	150.371	-1.2	46,198.2	45,905.4	208.512	-.055
5-24-33.....	175.965	100.057	150.128	0.7	70,205.6	70,101.8	208.591	.024
5-25-33.....	56.864	99.914	150.440	-1.3	45,741.6	45,257.2	208.571	.004
Group II. $Z_{100}^{150} = 33,397.0$ int. j. $\alpha_{100}^{150} = 208.567$ int. j/g = α_m .								
Even-temperature interval t_1 to $t_2 = 100$ to 150° . Combined mean value of $\alpha_{100}^{150} = 208.605$ int. j/g.								
1-17-33.....	45.854	150.153	199.449	-----	44,737.1	45,389.0	207.834	-.276
1-19-33.....	45.854	149.796	200.029	-----	45,586.3	45,379.9	207.635	-.475
1-23-33.....	218.231	149.363	200.055	-----	82,390.6	81,279.4	208.130	.020
1-24-33.....	218.231	149.743	199.488	-----	80,838.3	81,275.6	208.113	.003
1-25-33.....	218.231	150.168	199.659	-----	80,433.7	81,266.9	208.073	-.037
1-26-33.....	61.155	150.149	199.820	-----	48,276.4	48,598.0	208.307	.197
1-27-33.....	61.155	150.241	200.406	-----	48,791.5	48,611.7	208.531	.421
Group I. $Z_{150}^{200} = 35,859.0$ int. j. $\alpha_{150}^{200} = 208.110$ int. j/g = α_m .								

TABLE 1.—Principal data from a experiments—Continued

Date	Mass of water, M	Initial temperature	Final temperature	Heat leak	Entire energy added Q	Energy for even-temperature interval Q_H	α_j	Deviation α_j — α_m
	g	°C	°C	Int. j	Int. j	Int. j	Int. j/g	Int. j/g
5-9-33	221.000	149.766	199.851	—3.0	81,947.7	81,820.5	208.014	—0.123
5-10-33	125.970	150.037	200.019	—0.7	62,055.7	62,076.6	208.203	.066
5-11-33	59.989	150.143	200.110	—2.9	48,296.4	48,321.0	207.900	— .237
5-12-33	59.989	150.371	200.187	—1.2	48,168.2	48,330.1	208.051	— .086
5-23-33	253.166	150.158	200.341	0.7	88,908.2	88,569.5	208.244	.107
5-24-33	175.965	150.128	200.131	— .2	72,518.4	72,507.3	208.326	.189
5-25-33	56.864	150.440	200.140	—2.4	47,418.2	47,688.0	208.193	.056

Group II. $Z]_{150}^{200}=35,849.3$ int. j. $\alpha]_{150}^{200}=208.137$ int. j/g= α_m .
Even-temperature interval t_1 to $t_2=150$ to 200° . Combined mean value of $\alpha]_{150}^{200}=208.128$ int. j/g.

1-17-33	45.854	199.449	249.579	-----	48,874.9	48,786.3	206.545	— .496
1-19-33	45.854	200.029	250.725	-----	49,498.9	48,787.7	206.575	— .466
1-23-33	218.231	200.055	249.858	-----	84,146.3	84,453.8	206.975	— .066
1-24-33	218.231	199.488	249.738	-----	84,894.6	84,501.1	207.054	.013
1-25-33	218.231	199.659	249.850	-----	84,782.3	84,476.9	206.944	— .097
1-26-33	61.155	199.820	250.156	-----	52,342.7	51,993.1	207.304	.263
1-27-33	61.155	200.405	249.960	-----	51,540.9	51,991.8	207.283	.242
2-3-33	158.636	199.951	249.846	-----	72,046.4	72,205.3	207.329	.288

Group I. $Z]_{200}^{250}=39,315.4$ int. j. $\alpha]_{200}^{250}=207.041$ int. j/g= α_m .

5-9-33	221.000	199.851	250.158	— .6	85,538.6	85,014.7	206.819	— .182
5-10-33	125.970	200.019	249.921	—1.6	65,279.6	65,410.1	207.210	.209
5-11-33	59.989	200.110	250.109	—0.1	51,719.2	51,711.7	206.770	— .231
5-12-33	59.989	200.187	250.082	.4	51,616.4	51,714.5	206.816	— .185
5-23-33	253.166	200.341	249.825	—3.0	90,864.2	91,806.4	207.368	.367
5-24-33	175.965	200.131	250.011	—2.5	75,600.0	75,776.1	207.247	.246
5-25-33	56.864	200.141	249.918	—5.1	50,838.2	51,074.7	206.931	— .070

Group II. $Z]_{200}^{250}=39,307.8$ int. j. $\alpha]_{200}^{250}=207.001$ int. j/g= α_m .
Even-temperature interval t_1 to $t_2=200$ to 250° . Combined mean value of $\alpha]_{200}^{250}=207.014$ int. j/g.

1-17-33	45.854	249.579	299.461	-----	53,243.7	53,418.9	204.883	— .895
1-19-33	45.854	250.725	299.625	-----	52,268.2	53,426.9	205.057	— .721
1-23-33	218.231	249.858	299.708	-----	88,657.9	88,947.8	205.853	.075
1-24-33	218.231	249.738	300.137	-----	89,632.3	88,927.5	205.760	— .018
1-25-33	218.231	249.850	299.643	-----	88,513.3	88,908.4	205.673	— .105
1-26-33	61.155	250.156	300.022	-----	56,490.3	56,619.4	205.955	.177
1-27-33	61.155	249.960	299.818	-----	56,458.5	56,631.8	206.158	.380
2-6-33	158.636	250.276	300.285	-----	76,695.4	76,650.8	205.670	— .108
2-7-33	158.636	249.990	299.507	-----	75,914.8	76,683.5	205.876	.098
2-8-33	71.459	250.008	299.831	-----	58,545.5	58,763.0	206.255	.477
2-10-33	71.459	249.999	299.955	-----	58,691.6	58,746.1	206.019	.241
2-14-33	223.502	249.932	299.868	-----	89,891.3	90,016.7	205.781	.003

Group II. $Z]_{250}^{300}=44,024.2$ int. j. $\alpha]_{250}^{300}=205.778$ int. j/g= α_m .

5-10-33	125.970	249.921	299.914	—0.3	69,915.1	69,933.4	206.159	— .066
5-11-33	59.989	250.109	299.708	— .1	55,875.0	56,337.8	206.276	.051
5-12-33	59.989	249.764	299.848	—2.1	56,428.0	56,354.8	206.560	.335
5-24-33	175.965	250.011	299.623	—0.3	79,612.8	80,256.1	206.249	.024
5-25-33	56.864	249.918	300.076	—11.2	55,859.0	55,671.3	205.891	— .334

Group II. $Z]_{250}^{300}=43,963.5$ int. j. $\alpha]_{250}^{300}=206.225$ int. j/g= α_m .
Even-temperature interval t_1 to $t_2=250$ to 300° . Combined mean value of $\alpha]_{250}^{300}=206.076$ int. j/g.

1-26-33	61.155	300.022	325.087	-----	30,738.8	30,668.6	101.920	— .053
1-27-33	61.155	299.818	324.898	-----	30,744.1	30,656.8	101.727	— .246
2-2-33	61.220	299.965	324.993	-----	30,713.7	30,681.3	102.019	.046
2-6-33	158.636	300.285	325.389	-----	40,805.9	40,611.5	101.968	— .095
2-7-33	158.636	299.507	324.796	-----	41,067.2	40,622.8	102.039	.066
2-8-33	71.459	299.831	325.169	-----	32,159.2	31,728.7	102.059	.086
2-10-33	71.459	299.955	325.345	-----	32,236.0	31,727.5	102.042	.069
2-14-33	164.242	299.894	325.258	-----	41,817.7	41,210.8	102.136	.163
2-15-33	185.040	300.015	324.820	-----	42,947.7	43,291.5	101.901	— .072
2-16-33	176.362	299.880	325.047	-----	42,681.8	42,400.8	101.865	— .108

Group I. $Z]_{300}^{325}=24,435.7$ int. j. $\alpha]_{300}^{325}=101.973$ int. j/g= α_m .

TABLE 1.—Principal data from α experiments—Continued

Date	Mass of water, M	Initial temperature	Final temperature	Heat leak	Entire energy added Q	Energy for even temperature interval Q_1	α_1	Deviation $\alpha_1 - \alpha_m$
	g	°C	°C	Int. j	Int. j	Int. j	Int. j/g	Int. j/g
1-26-33	61.155	325.087	349.478	-----	32,110.1	32,937.1	101.390	-0.040
1-27-33	61.155	324.898	350.036	-----	33,123.2	32,943.6	101.496	.066
2-2-33	61.220	324.991	349.953	-----	32,887.5	32,940.0	101.330	-.100
2-6-33	158.636	325.389	349.758	-----	41,755.8	42,833.8	101.473	.043
2-7-33	158.636	324.796	349.757	-----	42,736.6	42,828.9	101.442	.012
2-8-33	71.459	325.169	350.221	-----	34,109.9	34,016.0	101.868	.438
2-10-33	71.459	325.345	349.811	-----	33,244.1	33,965.4	101.160	-.270
2-14-33	164.242	325.258	350.479	-----	43,765.8	43,338.3	101.081	-.349
2-15-33	185.040	324.820	349.738	-----	45,357.2	45,530.4	101.566	.136
2-16-33	176.362	325.047	349.691	-----	43,986.8	44,640.7	101.519	.089

Group I. $Z_{325}^{350} = 26,736.6$ int. j. $\alpha_{325}^{350} = 101.430$ int. j/g = α_m .

5-10-33	125.970	299.914	349.572	-0.7	76,186.6	76,765.7	204.204	.158
5-11-33	59.989	299.708	349.927	-.8	63,536.0	63,290.8	204.182	.136
5-12-33	59.989	299.848	349.973	-1.9	63,437.3	63,294.5	204.244	.198
5-24-33	175.965	299.623	349.756	2.5	87,109.9	86,935.1	203.978	-.068
5-25-33	56.864	300.086	349.933	-11.0	62,424.6	62,617.3	203.559	-.487

Group II. $Z_{300}^{350} = 51,042.1$ int. j. $\alpha_{300}^{350} = 204.046$ int. j/g = α_m .
Even-temperature interval t_1 to $t_2 = 300$ to 350° . Combined mean value of $\alpha_{300}^{350} = 203.832$ int. j/g.TABLE 2.—Principal data from β experiments

Date	Initial temperature	Approximate rate	Mass of liquid removed	Energy corrections	Entire energy added	Corrections to β	β at even temperature	Deviation $\beta - \beta_m$
	°C	g/min	g	Int. j/g	Int. j/g	Int. j/g	Int. j/g	Int. j/g
6-20-33	200.042	4.35	52.031	-48.0	972.8	-0.883	17.814	-0.115
6-20-33	200.063	9.52	171.678	1.1	3,210.6	-.824	17.877	-.052
6-21-33	200.072	9.88	227.568	-18.5	4,285.8	-.910	17.923	-.006
7-7-33	199.983	18.00	98.948	-67.0	1,728.8	-1.107	18.113	.184
7-10-33	200.024	5.13	102.556	-41.5	1,896.9	-0.580	17.916	-.013

Even temperature 200° . Mean value of $\beta_{200} = 17.929$ int. j/g.

7-17-33	230.016	9.70	117.472	-36.7	3,810.2	-1.166	31.269	-.025
7-18-33	230.018	9.72	194.442	-40.8	6,253.0	-0.860	31.302	.008
7-20-33	230.032	9.61	96.091	-50.4	3,093.0	-.876	31.312	.018

Even temperature 230° . Mean value of $\beta_{230} = 31.294$ int. j/g.

5-23-33	250.006	9.63	192.688	-9.4	8,630.2	-.747	44.041	-.020
6-6-33	249.989	4.83	193.835	-18.6	8,684.0	-.705	44.096	.035
6-19-33	249.978	4.74	94.972	-28.3	4,248.2	-.684	44.047	-.014
6-19-33	249.993	4.83	96.767	-1.2	4,319.4	-.576	44.061	.000

Even temperature 250° . Mean value of $\beta_{250} = 44.061$ int. j/g.

7-13-33	269.999	5.00	84.993	-17.2	5,224.4	-.466	61.003	-.039
7-14-33	270.003	4.95	84.370	-27.3	5,193.4	-.520	61.035	-.007
7-14-33	270.014	4.97	109.412	-7.0	6,741.2	-.549	61.064	.022
7-19-33	270.017	5.00	95.047	-47.3	5,852.4	-.508	61.066	.024

Even temperature 270° . Mean value of $\beta_{270} = 61.042$ int. j/g.

5-9-33	299.997	4.74	95.031	-10.1	9,281.7	-.329	97.341	-.113
5-26-33	299.990	4.64	148.481	-301.7	14,671.7	-1.283	97.529	.075
5-29-33	299.976	4.64	148.629	-20.3	14,534.1	-0.311	97.477	.023
7-11-33	299.992	4.75	161.357	21.9	15,781.0	-.332	97.470	.016

Even temperature 300° . Mean value of $\beta_{300} = 97.454$ int. j/g.

TABLE 3.—Principal data from γ experiments

Date	Initial temperature	Approximate rate	Mass of vapor removed	Energy corrections	Entire energy added	Correction to γ	γ at even temperature	Deviation $\gamma - \gamma_m$
	° C	g/min	g	<i>Int. j</i>	<i>Int. j</i>	<i>Int. j/g</i>	<i>Int. j/g</i>	<i>Int. j/g</i>
2-23-34	100.107	0.24	14.138	-15.6	31,912.3	0.28	2,257.49	-0.30
	99.992	1.18	48.369	-145.1	109,184.7	-0.02	2,257.30	-49
5-18-34	100.002	0.73	36.939	-22.3	83,439.9	.01	2,258.87	1.08
	100.023	1.40	41.871	-28.5	94,538.6	.06	2,257.91	0.12
	99.984	1.01	26.222	-65.7	59,212.7	-0.04	2,258.09	.30
5-23-34	100.036	1.01	26.246	19.1	59,260.6	.09	2,257.98	.19
	100.003	1.01	26.209	-21.3	59,179.3	.01	2,257.99	.20
	100.046	0.60	14.906	31.6	33,647.0	.12	2,257.40	-39
	99.997	.59	23.679	-21.8	53,450.8	-0.01	2,257.30	-49
5-28-34	100.005	.59	17.750	-18.6	40,071.0	.01	2,257.53	-26
	100.022	1.03	30.853	-37.3	69,660.7	.06	2,257.89	.10
	100.025	0.73	21.856	-2.8	49,336.6	.06	2,257.41	-38
5-29-34	100.032	1.27	25.329	-30.8	57,187.0	.08	2,257.85	.06

Even temperature 100°. Mean value of $\gamma_{100}=2,257.79$ int. j/g

5-31-34	150.004	1.19	35.629	-27.1	75,522.1	.01	2,119.69	.19
	150.026	1.48	44.475	-5.7	94,256.5	.08	2,119.39	-11
	150.030	1.48	44.429	-1.4	94,163.4	.09	2,119.50	.00
	150.002	.98	29.358	-24.7	62,225.7	.01	2,119.56	.06
6-1-34	150.023	0.58	17.566	1.8	37,234.2	.07	2,119.74	.24
	150.019	.58	17.551	-9.7	37,200.2	.06	2,119.61	.11
	150.021	1.75	52.456	-5.0	111,168.9	.06	2,119.34	-16

Even temperature 150°. Mean value of γ_{150} = 2,119.50 int. j/g

6-4-34-----	200.004	1.84	36.849	-16.4	72,115.5	.01	1,957.06	-.05
	200.014	1.84	36.793	-2.9	72,008.0	.05	1,957.16	.05
	200.016	0.53	15.819	2.0	30,951.4	.06	1,956.66	-.45
	200.014	0.53	15.812	-8.0	30,941.9	.05	1,956.91	-.20
6-5-34-----	199.991	1.43	28.504	-16.7	55,780.2	-.03	1,956.90	-.21
	200.002	1.43	28.485	2.4	55,754.7	.01	1,957.35	.24
	199.998	.93	18.676	-8.4	36,552.4	-.01	1,957.18	.07
	200.004	.93	37.345	6.3	73,097.4	.01	1,957.36	.25

Even temperature 200°. Mean value of $\gamma_{200}=1,957.11$ int. j/g

1-31-34	249.905	1.05	39.918	-11.2	70,192.9	-.42	1,758.12	-.62
	249.905	1.05	22.054	10.0	38,803.5	-.42	1,759.17	-.43
2-1-34	249.931	2.01	64.231	-9.2	112,917.3	-.31	1,757.67	-1.07
	249.931	2.01	60.170	17.2	105,838.0	-.31	1,758.73	-0.01
	249.948	0.52	14.581	-3.5	25,652.9	-.24	1,759.36	.62
	249.948	.52	15.622	2.9	27,476.3	-.24	1,758.84	.10
	249.948	.52	15.615	-0.4	27,472.4	-.24	1,759.38	.64
	249.948	.52	15.614	-2.5	27,464.3	-.24	1,758.97	.23
2-2-34	249.948	.52	15.606	-11.7	27,450.2	-.24	1,758.97	.23
	249.948	.52	15.598	-4.2	27,451.0	-.24	1,759.93	1.19
	249.948	.52	15.598	-2.6	27,448.1	-.24	1,759.73	0.99
	249.948	.52	15.599	7.5	27,454.2	-.24	1,760.02	1.28
	249.948	.52	15.602	-0.9	27,443.6	-.24	1,759.00	0.26
	250.002	2.04	40.791	5.8	71,725.7	.01	1,758.45	-.29
2-26-34	250.006	2.04	40.755	-4.4	71,638.8	.03	1,757.89	-.85
	250.005	2.04	40.711	2.3	71,585.0	.02	1,758.46	-.28
	250.007	2.04	30.508	5.2	53,659.0	.03	1,758.95	.21
	250.000	2.05	41.047	-20.6	72,189.6	.00	1,758.70	-.04
6-6-34	250.015	2.05	41.000	14.4	72,124.8	.07	1,759.21	.47
	250.007	0.52	20.778	-3.1	36,539.4	.03	1,758.59	-.15
	250.009	1.57	31.447	-0.8	55,316.4	.04	1,759.08	.34
	249.996	1.44	28.689	-8.4	50,469.9	-.02	1,759.19	.45
6-7-34	250.003	1.44	28.667	23.1	50,424.7	.01	1,758.99	.25
	249.986	.83	25.029	-18.0	44,023.5	-.06	1,758.84	.10

Even temperature 250°. Mean value of $\gamma_{250}=1,758.74$ int. j/g.

	270.021	1.02	20.376	5.0	33,918.1	.10	1,664.79	.08
	270.016	1.02	20.364	-2.5	33,905.6	.08	1,665.24	.53
2-9-34	270.014	1.02	20.364	-12.4	33,890.4	.07	1,664.48	— .23
	270.015	1.02	20.368	0.2	33,898.8	.07	1,664.57	— .14
	270.017	1.02	20.363	6.9	33,899.1	.09	1,665.01	.30
	270.018	1.02	20.369	6.3	33,892.8	.09	1,664.21	— .50
	270.007	2.03	40.690	-2.3	67,733.2	.04	1,664.75	.04
2-12-34	270.009	2.03	40.659	-2.6	67,680.6	.05	1,664.73	.02
	270.008	2.03	54.856	8.0	91,309.1	.04	1,664.65	— .06

Even temperature 270°. Mean value of $\gamma_{270}=1,664.71$ int. j/g.

TABLE 3.—Principal data from γ experiments—Continued

Date	Initial temperature	Approximate rate	Mass of vapor removed	Energy corrections	Entire energy added	Correction to γ	γ at even temperature	Deviation $\gamma - \gamma_m$
	° C	g/min	g	Int. J	Int. J	Int. J/g	Int. J/g	Int. J/g
3-6-34	350.025	2.03	40.538	-2.1	45,057.8	0.27	1,112.04	-0.34
	350.025	2.03	40.528	-3.3	45,038.7	.27	1,111.84	-.54
6-25-34	350.025	2.96	44.313	-16.4	49,256.3	.27	1,111.82	-.56
	350.032	1.82	36.292	19.6	40,352.4	.35	1,112.23	-.15
10-17-34	349.970	2.02	40.565	-18.7	45,136.6	-.33	1,112.37	-.01
	349.973	0.086	4.312	-22.3	4,797.8	-.29	1,112.37	-.01
	349.993	.48	4.768	-11.6	5,308.9	-.08	1,113.27	.89
1-15-35	349.996	3.21	32.110	-10.2	35,711.1	-.05	1,112.09	-.29
	349.997	0.061	1.820	-10.8	2,024.8	-.03	1,112.81	-.07
	349.996	1.08	10.814	-8.9	12,038.1	-.04	1,113.10	.72
	349.999	1.55	3.092	-7.0	3,438.1	-.01	1,112.10	-.28
	350.010	1.06	10.644	-10.1	11,847.3	.11	1,113.19	.81
1-16-35	350.016	0.50	4.967	-2.0	5,533.6	.17	1,114.20	1.82
	350.002	.063	1.893	-12.2	2,106.6	.03	1,112.87	0.49
	350.004	.18	3.589	-10.6	3,991.0	.05	1,112.06	-.32
	349.996	.48	4.766	-6.3	5,304.9	-.05	1,113.02	.64
1-18-35	349.993	.17	3.512	-9.7	3,904.8	-.08	1,111.73	-.65
	349.997	3.26	32.602	-8.4	36,261.1	-.03	1,112.22	-.16
	349.998	0.16	3.203	-6.4	3,565.1	-.02	1,113.06	.68
	349.998	1.01	10.152	0.5	11,301.0	-.02	1,113.10	.72
	349.991	0.17	3.365	-3.6	3,750.5	-.10	1,114.33	1.95
1-21-35	349.987	.17	3.360	-16.6	3,737.7	-.14	1,112.20	-0.18
	349.990	.17	3.358	-13.7	3,738.4	-.10	1,113.14	.76
	349.991	.17	3.362	-9.7	3,742.2	-.09	1,113.03	.65

Even temperature 350°. Mean value of $\gamma_{350}=1112.38$ int. J/g.

3-8-34	360.017	2.05	40.996	-12.0	40,507.6	.25	988.68	-.44
	360.022	2.04	38.939	-5.0	38,473.9	.32	988.72	-.40
3-9-34	360.002	2.96	29.664	-0.1	39,331.7	.03	989.17	.05
	360.008	3.69	40.631	-2.0	40,161.0	.12	988.72	-.40
3-12-34	360.000	3.69	36.902	-1.5	36,474.0	.00	988.57	-.55
	359.979	3.51	38.603	-11.6	38,191.4	-.31	989.03	-.09
10-23-34	359.979	0.038	2.033	-42.5	2,007.0	-.30	986.91	-2.21
	359.983	3.58	35.807	-8.1	35,425.8	-.25	989.10	-0.02
	359.982	0.038	1.968	-37.1	1,939.8	-.27	985.40	-3.72
10-24-34	359.985	3.60	36.043	-13.0	35,653.9	-.22	988.98	-0.14
	359.985	0.11	3.400	-14.1	3,375.9	-.22	992.69	3.57
	360.010	.55	10.925	-10.1	10,810.4	.14	989.61	0.49
1-10-35	360.011	3.66	36.636	-6.8	36,242.3	.16	989.42	.30
	360.014	0.074	2.208	-2.8	2,185.6	.21	990.02	.90
	360.017	.18	3.638	0.9	3,697.8	.25	989.17	.05
1-11-35	360.001	.067	2.008	-11.3	1,995.3	.02	993.50	4.38
	360.006	3.66	36.594	-3.2	36,192.4	.08	989.12	0.00
	360.008	0.54	5.430	1.6	5,371.4	.11	989.37	.25
1-22-35	359.993	.34	6.829	-2.8	6,765.7	-.10	990.63	1.51
	359.995	.34	6.824	-3.4	6,757.6	-.07	990.26	1.14
	359.999	.34	6.827	1.8	6,759.2	-.02	990.03	0.91
	359.998	.34	6.825	-0.2	6,750.2	-.04	989.01	-.11
1-24-35	360.002	1.87	18.730	-11.8	18,533.6	.03	989.55	.43
	360.006	1.87	18.697	-1.9	18,494.9	.10	989.31	.19
	360.007	1.87	18.710	-0.5	18,502.0	.10	989.00	-.12
	360.010	1.87	18.704	3.7	18,511.1	.15	989.82	.70

Even temperature 360°. Mean value of $\gamma_{360}=989.12$ int. J/g.

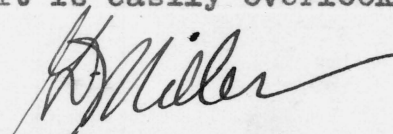
3-13-34	365.003	3.95	39.522	6.5	35,925.6	.06	909.26	-.25
	365.002	3.95	27.646	-10.2	25,116.2	.03	908.72	-.79
3-19-34	365.001	0.98	19.601	-10.1	17,815.5	.02	909.63	.12
	365.002	.98	19.626	-14.4	17,745.7	.04	909.56	.05
3-21-34	365.009	.49	9.839	-4.4	8,932.2	.16	909.37	-.14
	365.010	.97	14.670	-9.3	13,232.4	.18	909.08	-.43
	365.014	.97	14.558	-7.0	13,226.5	.25	909.50	-.01
1-2-35	365.016	.39	39.351	1.7	35,783.2	.30	909.63	.12
	365.014	.07	1.402	-10.8	1,281.5	.26	914.31	4.80
	365.021	.58	5.841	-12.9	5,314.8	.38	910.29	0.78
1-3-35	365.007	.58	5.809	-28.3	5,257.5	.13	910.36	.85
	365.021	.17	1.733	-26.6	1,582.0	.38	913.25	3.74
	365.001	.57	5.752	-7.0	5,241.4	.02	911.25	1.74
1-8-35	365.007	3.87	30.970	-3.3	28,160.6	.12	909.41	-0.10
	365.013	0.078	2.332	-4.2	2,125.4	.23	911.64	2.13

Even temperature 365°. Mean value of $\gamma_{365}=909.51$ int. J/g

There are a number of blank pages in the April 1937 issue of the Journal of Research of the National Bureau of Standards.

It is our practice to start each new article in the Journal with a facing page and this accounts for pages 448, 466, and 498 being blank. Page 443 was made blank so as to have Table 5 on pages facing each other.

A number of inquiries have been received concerning this issue of the Journal, all of which inquired about Table 4 and page 443. Table 4 will be found on page 418; as it is a very small table it is easily overlooked.



G.D. Miller, O-7, Information Sec.

TABLE 5.—*Formulated Data—Saturated Steam—International joules per gram*

Temperature °C	Latent heat $\gamma - \beta$	$\frac{1}{2}(\gamma + \beta)$	β	γ	$\alpha)_s^t$	Enthalpy liquid $H_l - H_s$	Enthalpy vapor $H_v - H_s$	Tdp/dT Int J/cm ²
100	2256.77		1.41	2258.17	417.36	418.76	2675.52	1.349 38
105	2243.41		1.66	2245.06	438.21	439.85	2683.26	1.581 33
110	2229.87		1.94	2231.81	459.06	460.99	2690.85	1.843 94
115	2216.13		2.26	2218.39	479.91	482.16	2698.29	2.139 95
120	2202.20	From 330 to 374.15° C values from equation $\frac{1}{2}(\gamma + \beta) = 535.08 + 18.413(374.15 - t)^{.617}$ From 100 to 330° C values of β are from Smith and Keyes liquid specific-volume data.	2.62	2204.82	500.76	503.37	2705.56	2.472 21
125	2188.04		3.03	2191.07	521.62	524.63	2712.68	2.843 70
130	2173.67		3.48	2177.16	542.47	545.94	2719.62	3.257 45
135	2159.07		3.99	2163.06	563.32	567.30	2726.37	3.716 59
140	2144.22		4.56	2148.78	584.18	588.73	2732.95	4.224 40
145	2129.12		5.19	2134.31	605.03	610.21	2739.32	4.784 20
150	2113.74		5.89	2119.63	625.88	631.76	2745.49	5.399 36
155	2098.08		6.66	2104.74	646.73	653.37	2751.45	6.073 38
160	2082.12		7.50	2089.62	667.57	675.07	2757.18	6.809 75
165	2065.83		8.44	2074.27	688.41	696.84	2762.67	7.612 11
170	2049.17		9.45	2058.63	709.25	718.69	2767.87	8.484 10
175	2032.10		10.57	2042.67	730.08	740.64	2772.74	9.429 51
180	2014.61		11.78	2026.39	750.91	762.68	2777.29	10.452 1
185	1996.67		13.11	2009.78	771.72	784.82	2781.49	11.555 6
190	1978.28		14.55	1992.83	792.53	807.07	2785.35	12.744 1
195	1959.42		16.11	1975.53	813.33	829.43	2788.86	14.021 3
200	1940.07		17.80	1957.88	834.13	851.91	2791.99	15.391 3
205	1920.22		19.63	1939.85	854.91	874.53	2794.75	16.858 5
210	1899.84		21.61	1921.45	875.68	897.27	2797.11	18.426 5
215	1878.91		23.74	1902.65	896.43	920.16	2799.07	20.099 8
220	1857.41		26.04	1883.45	917.18	943.21	2800.62	21.882 6
225	1835.31		28.52	1863.82	937.91	966.41	2801.72	23.779
230	1812.58		31.18	1843.76	958.63	989.80	2802.38	25.794
235	1789.20		34.04	1823.24	979.33	1013.36	2802.56	27.932
240	1765.13		37.12	1802.24	1000.02	1037.13	2802.25	30.198
245	1740.33		40.42	1780.75	1020.69	1061.10	2801.43	32.596
250	1714.77		43.96	1758.73	1041.35	1085.30	2800.07	35.132

255	1688.40		47.76	1736.16	1061.99	1109.74	2798.14	37.811
260	1661.17		51.83	1713.01	1082.61	1134.44	2795.61	40.638
265	1633.03		56.21	1689.24	1103.22	1159.41	2792.44	43.619
270	1603.92		60.90	1664.81	1123.81	1184.69	2788.61	46.760
275	1573.75		65.93	1639.68	1144.38	1210.29	2784.05	50.069
280	1542.47		71.33	1613.81	1164.93	1236.25	2778.72	53.550
285	1509.98		77.14	1587.12	1185.46	1262.59	2772.57	57.213
290	1476.17		83.38	1559.55	1205.97	1289.35	2765.52	61.064
295	1440.92		90.11	1531.03	1226.47	1316.57	2757.49	65.113
300	1404.09		97.36	1501.45	1246.94	1344.29	2748.38	69.397
305	1365.50		105.20	1470.70	1267.40	1372.58	2738.08	73.838
310	1324.92		113.68	1438.60	1287.83	1401.50	2726.42	78.537
315	1282.19		122.89	1405.08	1308.25	1431.13	2713.32	83.475
320	1237.24		132.93	1370.18	1328.64	1461.56	2698.81	88.667
325	1189.75		143.92	1333.67	1349.02	1492.93	2682.68	94.130
330	1139.29	725.65	156.00	1295.29	1369.37	1525.36	2664.65	99.882
335	1085.29	712.03	169.39	1254.67	1389.71	1559.08	2644.36	105.946
340	1027.00	697.72	184.22	1211.22	1410.02	1594.23	2621.23	112.355
345	963.38	682.58	200.89	1164.27	1430.31	1631.19	2594.57	119.145
350	892.85	666.42	219.99	1112.84	1450.58	1670.56	2563.42	126.37
355	812.98	648.90	242.41	1055.39	1470.83	1713.23	2526.21	134.10
360	719.43	629.52	269.81	989.23	1491.06	1760.86	2480.28	142.46
365	603.25	607.24	305.62	908.87	1511.27	1816.88	2420.12	151.60
366	575.69	602.27	314.42	890.12	1515.31	1829.72	2405.42	153.55
367	546.04	597.06	324.04	870.08	1519.35	1843.38	2389.42	155.55
368	513.79	591.56	334.66	848.46	1523.39	1858.04	2371.84	157.6
369	478.24	585.70	346.58	824.82	1527.42	1873.99	2352.23	159.7
370	438.30	579.39	360.24	798.54	1531.46	1891.69	2329.99	161.9
371	392.10	572.45	376.40	768.50	1535.49	1911.88	2303.98	164.2
372	336.04	564.61	396.59	732.63	1539.53	1936.11	2272.15	166.6
373	260.98	555.15	424.66	685.64	1543.56	1968.21	2229.19	169.1
374	114.61	540.79	483.49	598.10	1547.59	2031.07	2145.67	172.4
374.15	0.00	535.08	535.08	535.08	1548.20	2083.27	2083.27	173

TABLE 6.—*Properties of saturated steam*

Temperature °C	Vapor pressure kg/cm ²	Enthalpy (International Steam Table cal/g)			Specific Volume (cm ³ /g)		Entropy (IT cal/g°C)	
		Liquid	Latent heat	Vapor	Liquid	Vapor	Liquid	Vapor
100	1.03323	100.04	539.12	639.2		1673.5	0.3120	1.7567
105	1.2318	105.08	535.93	641.0		1419.7	.3254	1.7426
110	1.4609	110.12	532.69	642.8		1210.4	.3386	1.7289
115	1.7239	115.18	529.41	644.6		1036.7	.3517	1.7156
120	2.0425	120.25	526.08	646.3		891.8	.3647	1.7028
125	2.3666	125.33	522.70	648.0		770.5	.3775	1.6903
130	2.7544	130.42	519.27	649.7		668.36	.3902	1.6782
135	3.1923	135.52	515.78	651.3		582.00	.4027	1.6664
140	3.6848	140.64	512.23	652.9		508.66	.4151	1.6549
145	4.2369	145.77	508.62	654.4		446.12	.4275	1.6438
150	4.8535	150.92	504.95	655.9		392.57	.4397	1.6329
155	5.5402	156.08	501.21	657.3		346.55	.4518	1.6224
160	6.3023	161.27	497.40	658.7		306.86	.4637	1.6120
165	7.1457	166.47	493.50	660.0		272.50	.4756	1.6019
170	8.0764	171.69	489.52	661.2		242.65	.4874	1.5920
175	9.1006	176.93	485.45	662.4		216.63	.4991	1.5823
180	10.2225	182.20	481.27	663.5		193.87	.5107	1.5728
185	11.456	187.49	476.98	664.5		173.92	.5223	1.5634
190	12.800	192.80	472.59	665.4		156.37	.5337	1.5541
195	14.265	198.14	468.08	666.2		140.90	.5451	1.5450
200	15.857	203.51	463.46	667.0		127.21	.5564	1.5360
205	17.585	208.92	458.72	667.6		115.07	.5677	1.5270
210	19.456	214.35	453.85	668.2		104.28	.5789	1.5182
215	21.477	219.82	448.85	668.7		94.66	.5900	1.5095
220	23.659	225.32	443.71	669.0		86.07	.6011	1.5009
225	26.007	230.87	438.43	669.3		78.380	.6122	1.4923
230	28.531	236.45	433.01	669.5		71.479	.6232	1.4838
235	31.239	242.08	427.42	669.5		65.274	.6342	1.4753
240	34.140	247.76	421.67	669.4		59.681	.6451	1.4669

Below 330° C Smith and Keyes liquid specific volumes used in preference to calorimetric values. Above 330° C values of liquid specific volumes from calorimetric data for β are preferred.

245	37. 244	253. 49	415. 75	669. 2		54. 631	. 6561	1. 4584
250	40. 560	259. 27	409. 64	668. 9		50. 061	. 6670	1. 4500
255	44. 097	265. 10	403. 34	668. 4		45. 917	. 6779	1. 4416
260	47. 866	271. 00	396. 84	667. 8		42. 153	. 6888	1. 4331
265	51. 875	276. 97	390. 11	667. 1		38. 727	. 6997	1. 4246
270	56. 137	283. 01	383. 16	666. 2		35. 603	. 7106	1. 4161
275	60. 660	289. 13	375. 95	665. 1		32. 749	. 7216	1. 4074
280	65. 457	295. 33	368. 48	663. 8		30. 136	. 7326	1. 3987
285	70. 539	301. 62	360. 72	662. 3		27. 741	. 7436	1. 3899
290	75. 917	308. 01	352. 64	660. 7		25. 540	. 7547	1. 3809
295	81. 603	314. 52	344. 22	658. 7		23. 513	. 7659	1. 3718
300	87. 611	321. 14	335. 42	656. 6		21. 645	. 7772	1. 3624
305	93. 952	327. 90	326. 20	654. 1		19. 918	. 7885	1. 3528
310	100. 64	334. 80	316. 51	651. 3		18. 317	. 8000	1. 3428
315	107. 69	341. 88	306. 30	648. 2		16. 832	. 8117	1. 3325
320	115. 12	349. 15	295. 56	644. 7		15. 453	. 8236	1. 3219
325	122. 95	356. 64	284. 22	640. 9		14. 168	. 8357	1. 3109
330	131. 18	364. 39	272. 16	636. 6	1. 562	12. 968	. 8481	1. 2993
335	139. 85	372. 45	259. 26	631. 7	1. 599	11. 842	. 8609	1. 2872
340	148. 96	380. 85	245. 34	626. 2	1. 640	10. 780	. 8741	1. 2742
345	158. 54	389. 68	230. 14	619. 8	1. 686	9. 77	. 8878	1. 2601
350	168. 63	399. 08	213. 29	612. 4	1. 741	8. 81	. 9023	1. 2446
355	179. 24	409. 27	194. 21	603. 5	1. 808	7. 87	. 9179	1. 2271
360	190. 42	420. 65	171. 86	592. 5	1. 894	6. 94	. 9352	1. 2066
365	202. 21	434. 03	144. 11	578. 1	2. 016	6. 00	. 9554	1. 1812
366	204. 64	437. 10	137. 53	574. 6	2. 048	5. 80	. 9600	1. 1752
367	207. 11	440. 36	130. 44	570. 8	2. 083	5. 59	. 9649	1. 1687
368	209. 60	443. 87	122. 74	566. 6	2. 124	5. 38	. 9702	1. 1616
369	212. 12	447. 68	114. 25	561. 9	2. 170	5. 16	. 9759	1. 1538
370	214. 68	451. 9	104. 71	556. 6	2. 225	4. 93	. 9823	1. 1451
371	217. 26	456. 7	93. 7	550	2. 293	4. 68	. 9896	1. 1350
372	219. 88	462. 5	80. 3	542	2. 38	4. 40	. 9983	1. 1228
373	222. 53	470. 2	62. 3	533	2. 51	4. 05	1. 0100	1. 1065
374	225. 22	485. 2	27. 4	513	2. 80	3. 47	1. 0329	1. 0752
374.15	225. 65	498	0	498	3. 1	3. 1	1. 0521	1. 0521